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and Vehicle Engineering**



# **EXPERIMENTAL METHODS**

## **FOR IN-DUCT AEROACOUSTIC MEASUREMENTS**

**Hans Bodén**

# CONTENTS



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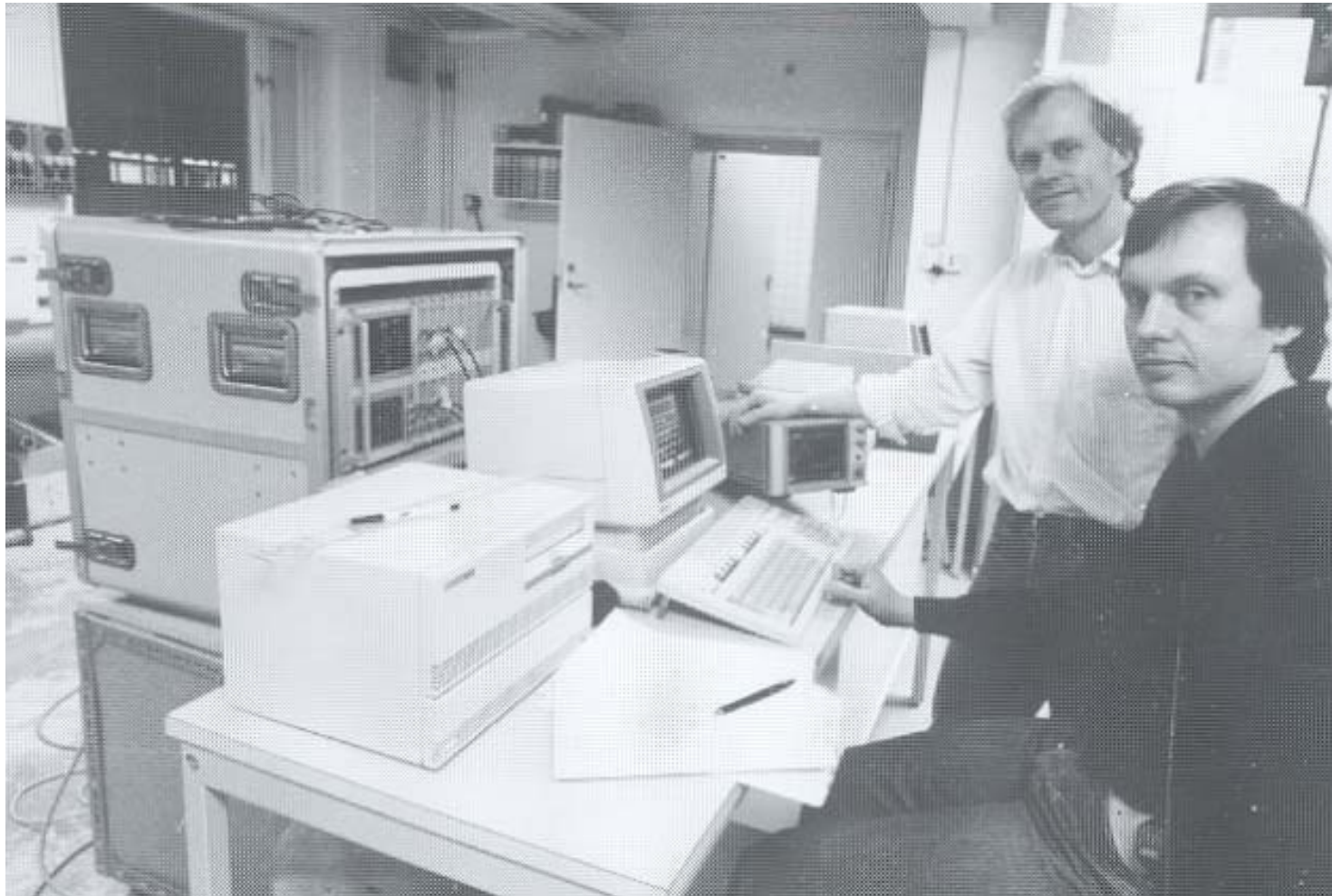
- Introduction.
- Passive one-port measurement techniques  
The two-microphone technique.  
Sources of errors.  
Effect of test rig design.
- Passive two-port measurement techniques.  
Flow noise suppression
- Application to liner impedance measurements
- Summary

# INTRODUCTION



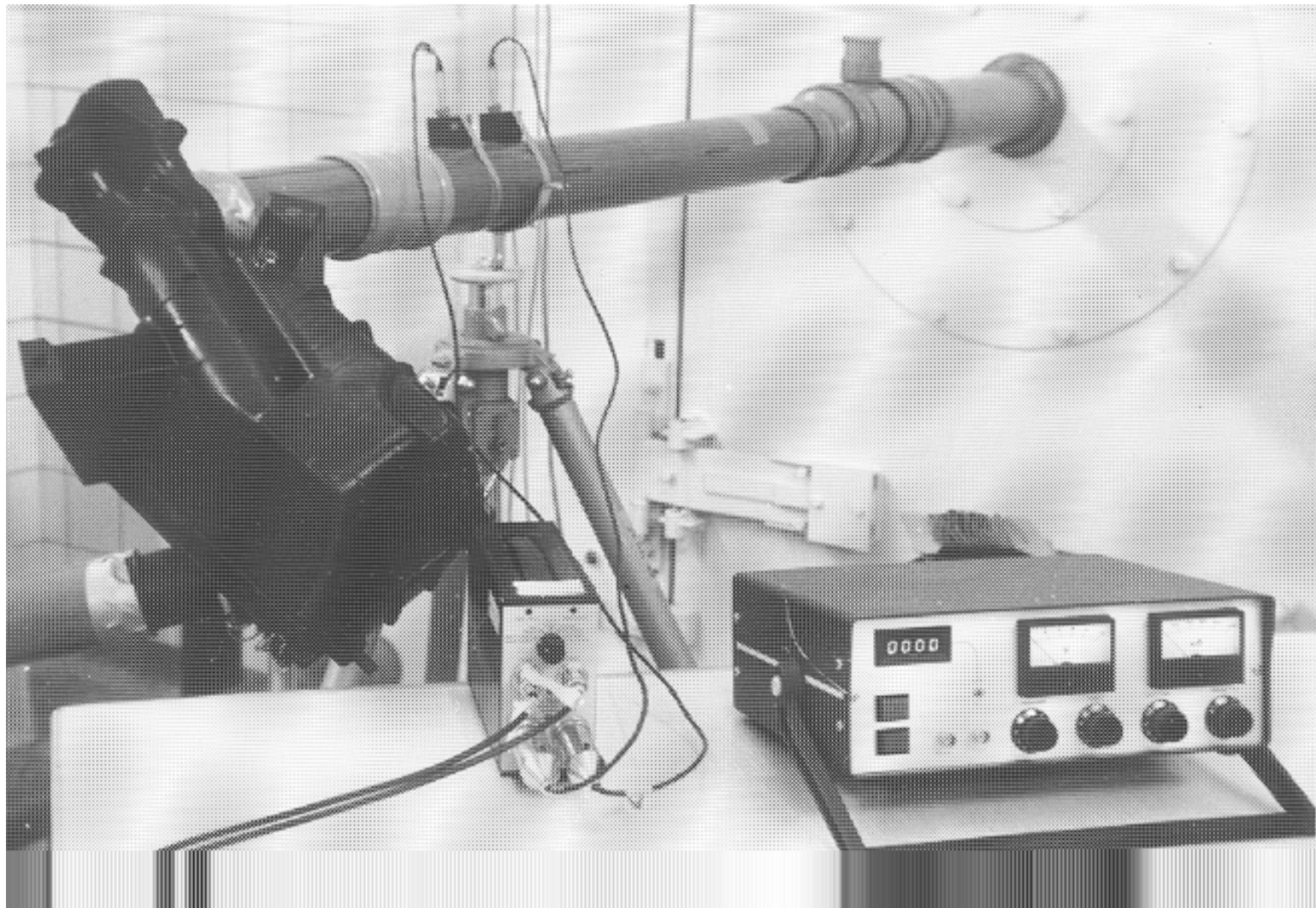
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**1986?**

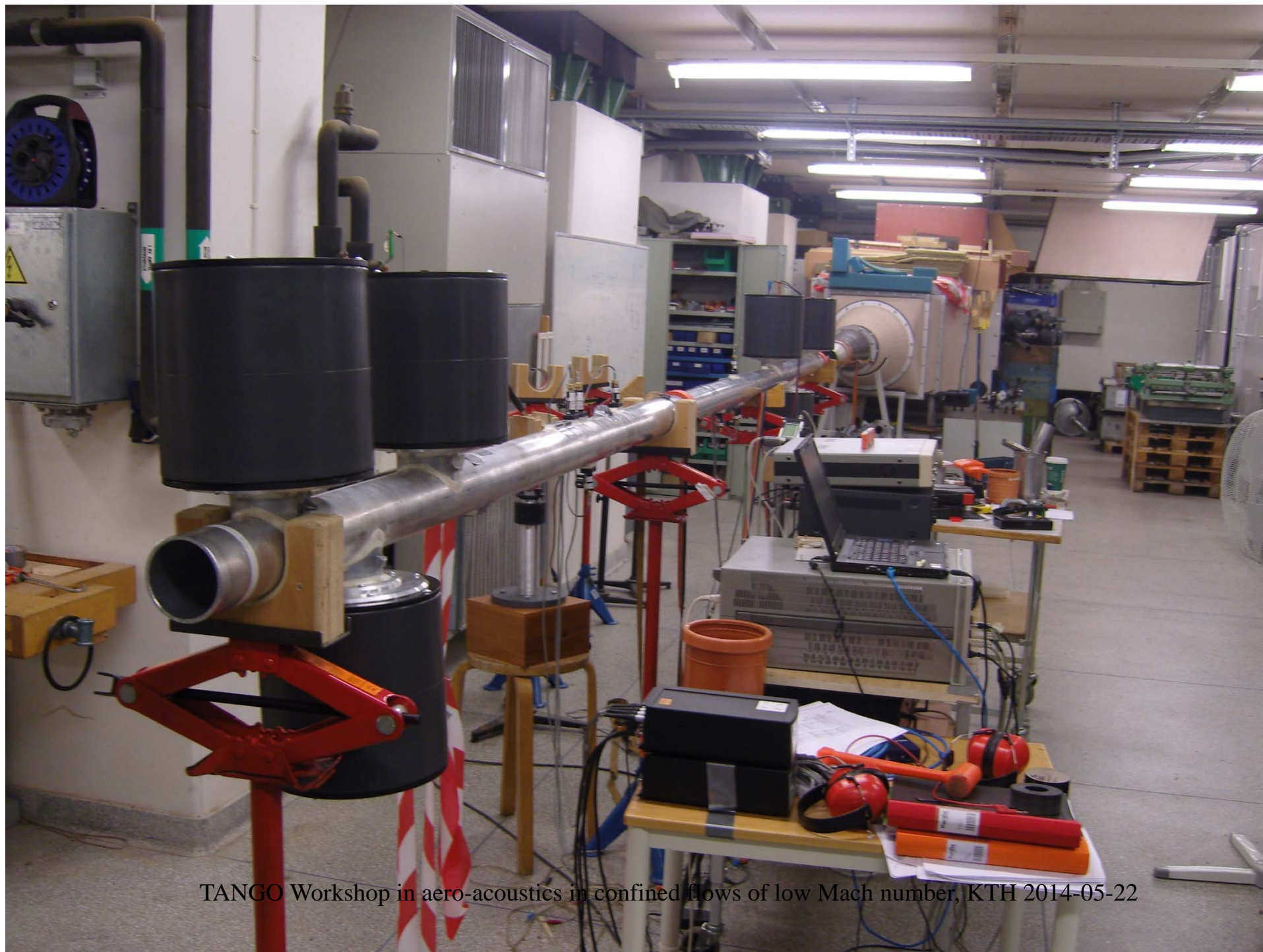




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TANGO Workshop in aero-acoustics in confined flows of low Mach number, KTH 2014-05-22

# **Influence of errors on the two-microphone method for measuring acoustic properties in ducts**

Hans Bodén and Mats Åbom

*Department of Technical Acoustics, Royal Institute of Technology, 100 44 Stockholm, Sweden*

(Received 6 December 1984; accepted for publication 9 September 1985)

Using the two-microphone method, acoustic properties in ducts, as, for example, reflection coefficient and acoustic impedance, can be calculated from a transfer function measurement between two microphones. In this paper, a systematic investigation of the various measurement errors that can occur and their effect on the calculated quantities is made. The input data for the calculations are the measured transfer function, the microphone separation, and the distance between one microphone and the sample. First, errors in the estimate of the transfer function are treated. Conclusions concerning the most favorable measurement configuration to avoid these errors are drawn. Next, the length measurement errors are treated. Measurements were made to study the question of microphone interference. The influence of errors on the calculated quantities has been investigated by numerical simulation. From this, conclusions are drawn on the useful frequency range for a given microphone separation and on the magnitude of errors to expect for different cases.

PACS numbers: 43.85.Vb, 43.85.Bh, 43.88.Kb, 43.55.Ev

541

J. Acoust. Soc. Am. 79 (2), February 1986 0001-4966/86/020541-09\$00.80

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541



# Error analysis of two-microphone measurements in ducts with flow

Mats Åbom and Hans Bodén

*Department of Technical Acoustics, Royal Institute of Technology, S-100 44 Stockholm, Sweden*

(Received 2 February 1987; accepted for publication 18 January 1988)

In an earlier work [H. Bodén and M. Åbom, *J. Acoust. Soc. Am.* **79**, 541–549 (1986)] the influence of errors on two-microphone measurements in ducts without flow has been studied. The aim of this article is mainly to extend the earlier work to include the effects of mean flow and also of attenuation during the sound propagation. First, a short review of the various existing two-microphone methods is made. The errors in the measured input data are then analyzed and special attention is paid to the effects of neglected attenuation, nonideal microphones, and flow noise. The influence of errors on the calculated quantities has been investigated and the conclusions from the earlier work have been extended to the case with flow. It is also shown that the neglect of attenuation between the microphones leads to a low-frequency limit for the applicability of the two-microphone method. Finally, a new technique for measuring the Mach number using a two-microphone method is suggested.

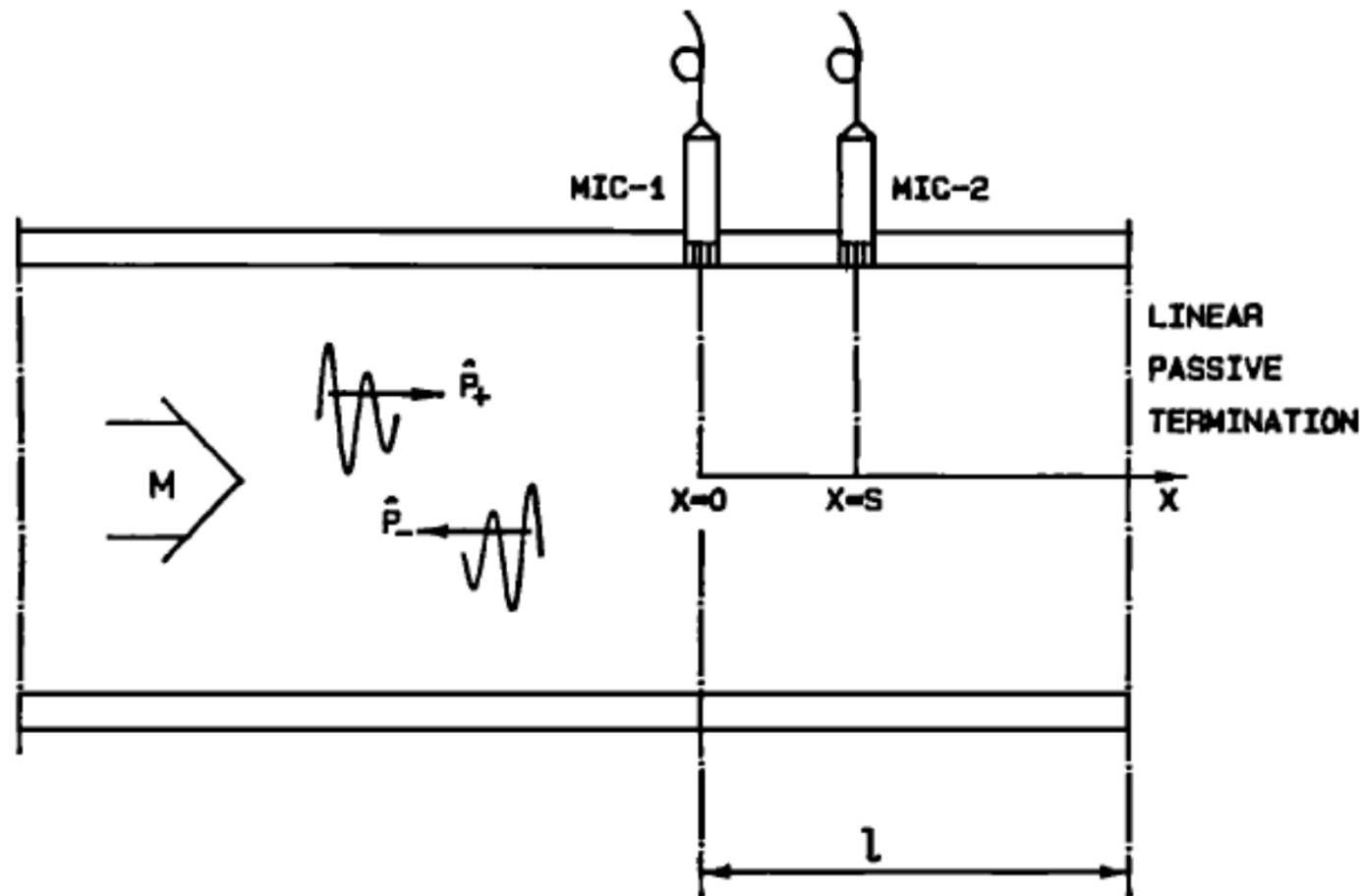
PACS numbers: 43.88.Kb, 47.60. + i, 43.85.Vb, 43.50.Nm

# Passive one-port measurement techniques

## The "Two-Microphone Method"



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## The Two-Microphone Method

$$p_1(f) = p_+(f) + p_-(f)$$

$$p_2(f) = p_+(f) \cdot \exp(-jk_+s) + p_-(f) \cdot \exp(jk_-s)$$



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where

$$k_+ = \frac{k}{1+M} \quad k_- = \frac{k}{1-M}$$

A linear system of equations in  $p_+$  and  $p_-$  from which the **Reflection Coefficient** at  $x=0$  can be calculated

$$R_0(f) = \frac{p_-(f)}{p_+(f)} = \frac{\exp(-jk_+s) - p_2(f)/p_1(f)}{p_2(f)/p_1(f) - \exp(jk_-s)}$$

# The Two-Microphone Method

With  $H_{12}(f) = \frac{p_2(f)}{p_1(f)}$  we get

$$R_0(f) = \frac{\exp(-jk_+s) - H_{12}(f)}{H_{12}(f) - \exp(jk_-s)}$$



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The reflection coefficient at  $x=l$  can be calculated from

$$R_l(f) = R_0(f) \frac{\exp(j2kl)}{1 - M^2}$$

And the normalised impedance (= one port passive system properties) can be calculated from

$$Z(f) = \frac{p(f)}{\rho_0 c \cdot u(f)} = \frac{1 + R(f)}{1 - R(f)}$$

## Errors in the Two-Microphone Method

$$R_0(f) = \frac{\exp(-jk_+s) - H_{12}(f)}{H_{12}(f) - \exp(jk_-s)}$$



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**Errors in the input data:  $k_+s$ ,  $k_-s$  and  $H_{12}(f)$**

**Errors in  $k_+s$  and  $k_-s$ :**

- Uncertainty in determination of  $k$  because of mainly turbulent losses
- Uncertainty in Mach-number measurement
- Uncertainty in length measurement: geometric and acoustic length



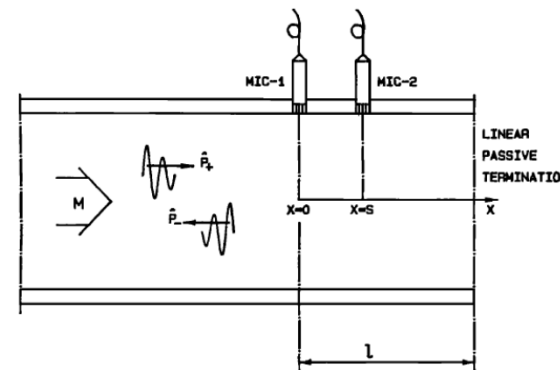
# Errors in the Two-Microphone Method

## Errors in $H_{12}(f)$

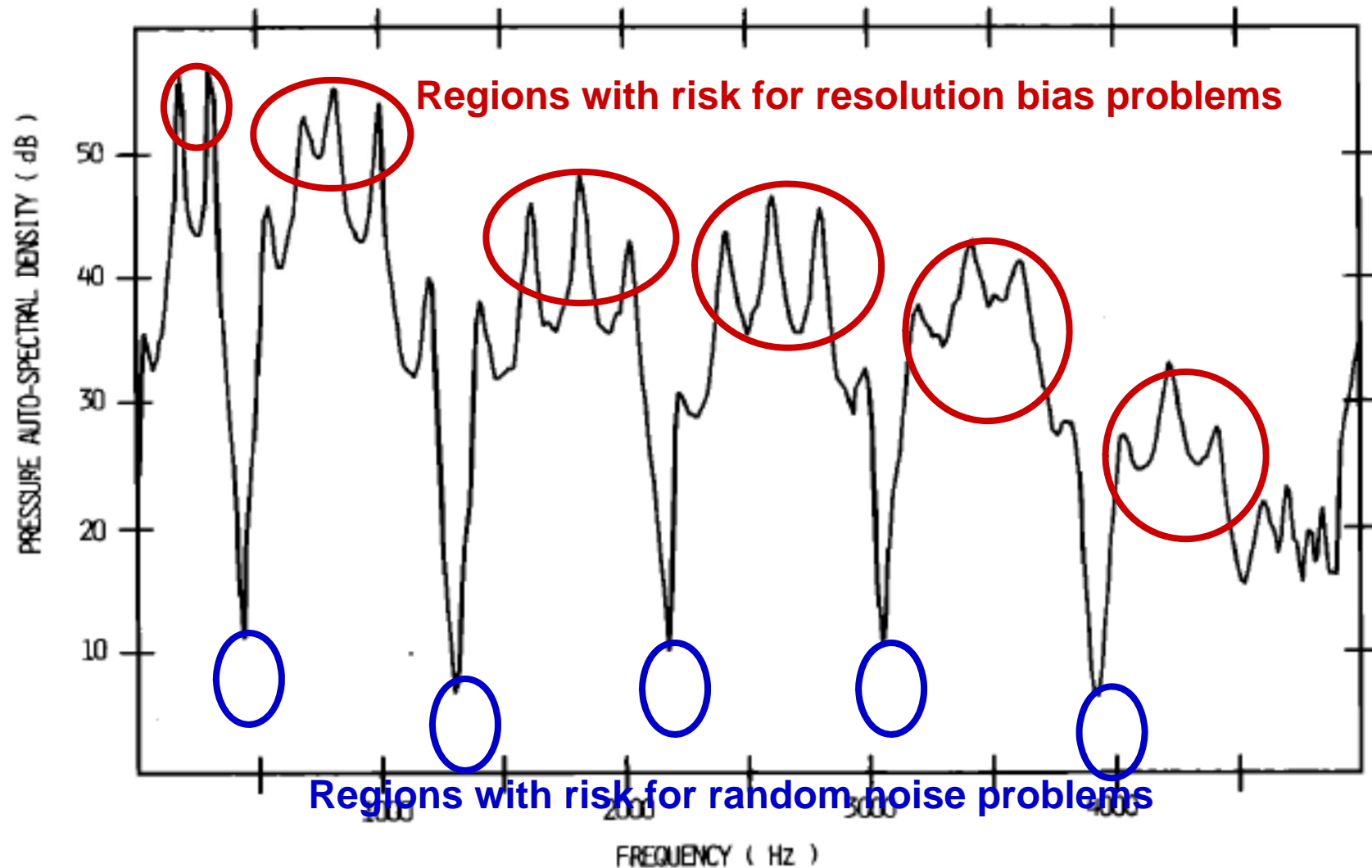


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- **Bias errors**, as for instance resolution-bias errors. Problem for long duct systems with many resonances. **Solution – Reflection free terminations.**
- **Random errors** caused by random signals but mainly flow noise disturbances



**Pressure autospectral density measured in a duct driven by a loudspeaker and with a rigid termination.**



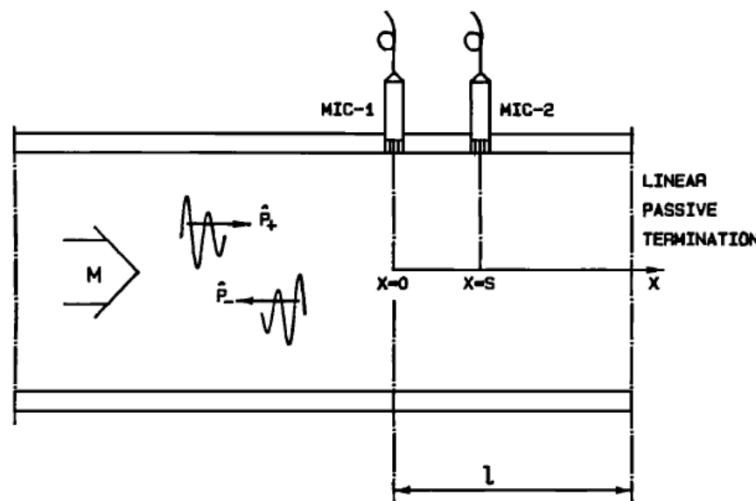
## Errors in the Two-Microphone Method

To avoid large sensitivity to the errors in the input data the two-microphone technique should be restricted to the frequency range:

$$0.1 \cdot \pi \cdot (1 - M^2) < ks < 0.8 \cdot \pi \cdot (1 - M^2)$$



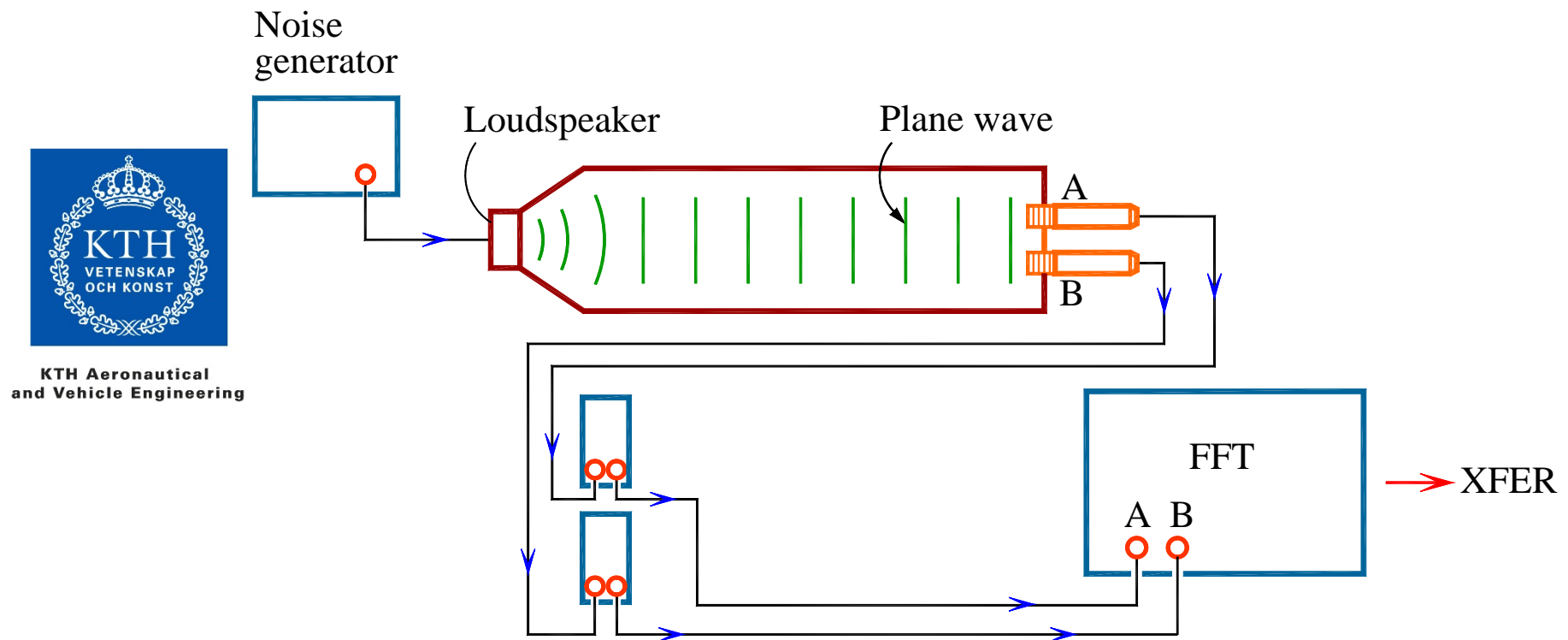
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# Calibration

Duct method for measurement of  $K_{AB}(\omega)$ .



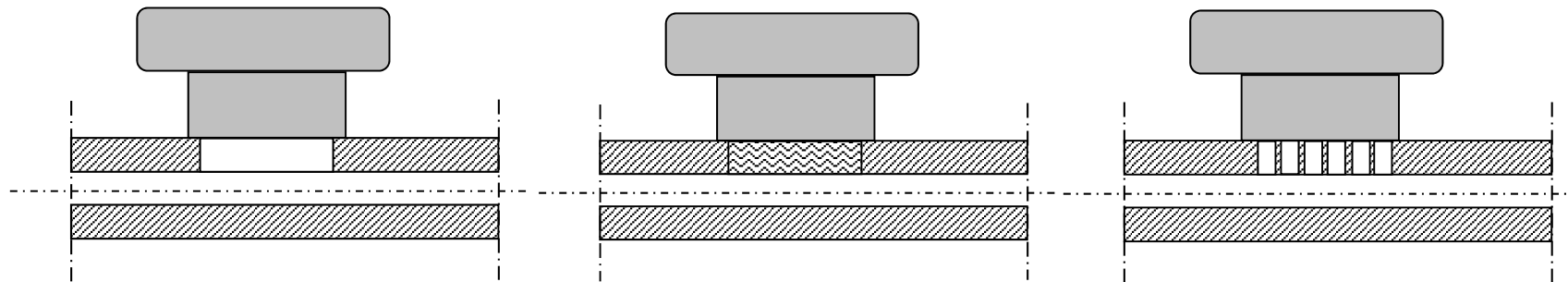


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TANGO Workshop in  
aero-acoustics in  
confined flows of low  
Mach number, KTH  
2014-05-22

# Effect of loudspeaker mounting configurations



a) loudspeaker connected using an open hole.

b) the hole filled with absorbing material

c) loudspeaker connected using a perforate pipe with 50% porosity.

Tests at  $M = 0.15$  and  $0.3$  showed that configuration c) with a 50% perforate gave the best signal-to-noise ratio at the microphones

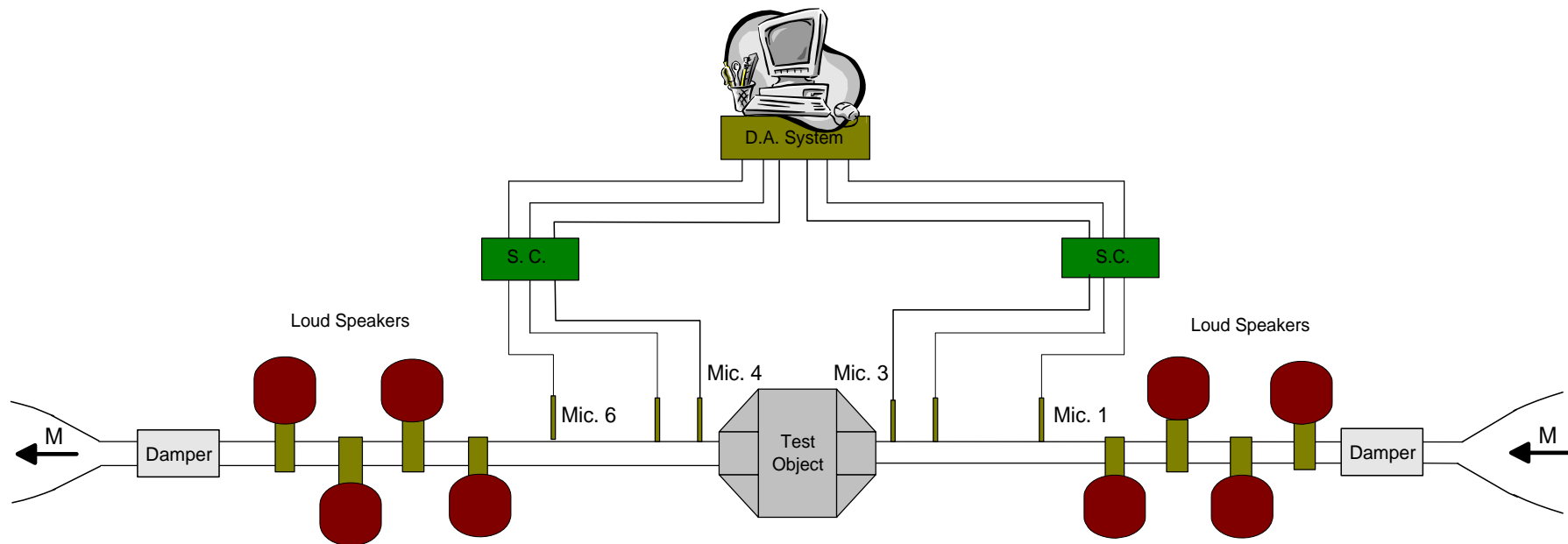
Signal to-noise ratio

$P_S$  is the sound power of the acoustic signal and  
 $P_N$  is the unwanted noise sound power.

$$SNR = 10 \log_{10} \left( \frac{P_S}{P_N} \right)$$

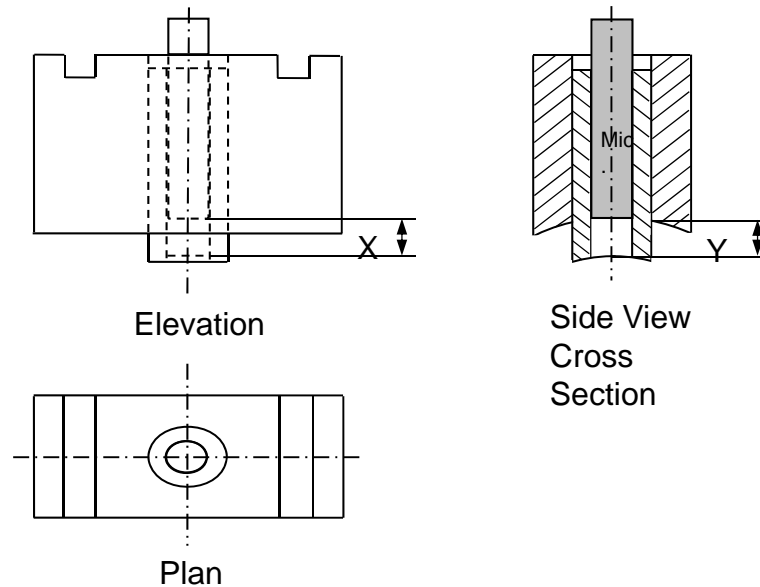


# Effect of loudspeaker mounting configurations

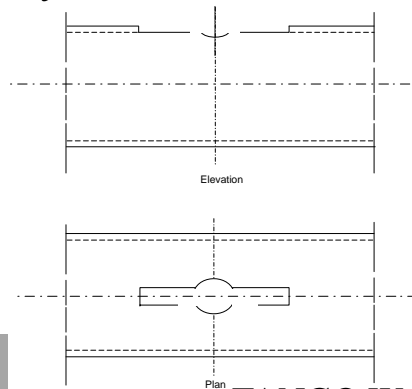


One should also avoid equidistant loudspeaker separations which may cause cancellation at certain frequencies

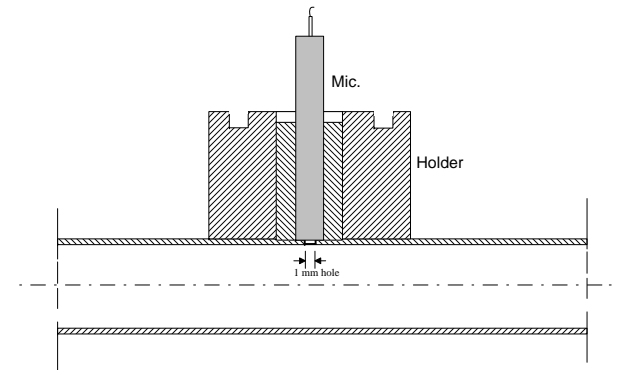
# Effect of microphone holders



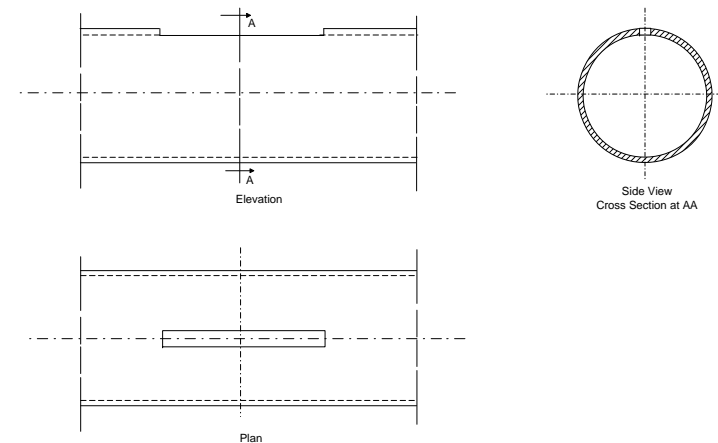
Reference holder with flush mounted microphone. Also used with the microphone moved 1.5 mm and 3 mm away from the duct wall



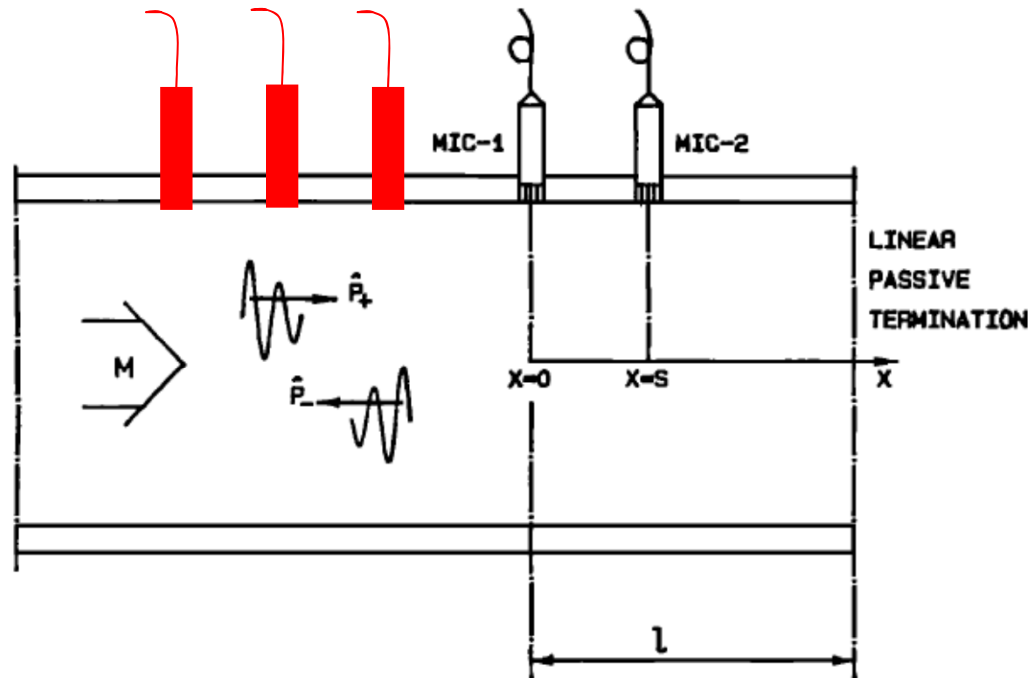
**Microphone connected to duct via a 30 mm long and 2 mm wide slit. Also a 6.4 mm diameter hole added to the slit.**



Microphone connected to duct via a 1 mm diameter hole.



## Over-determination



Add more  
microphones

Can be used to  
reduce error

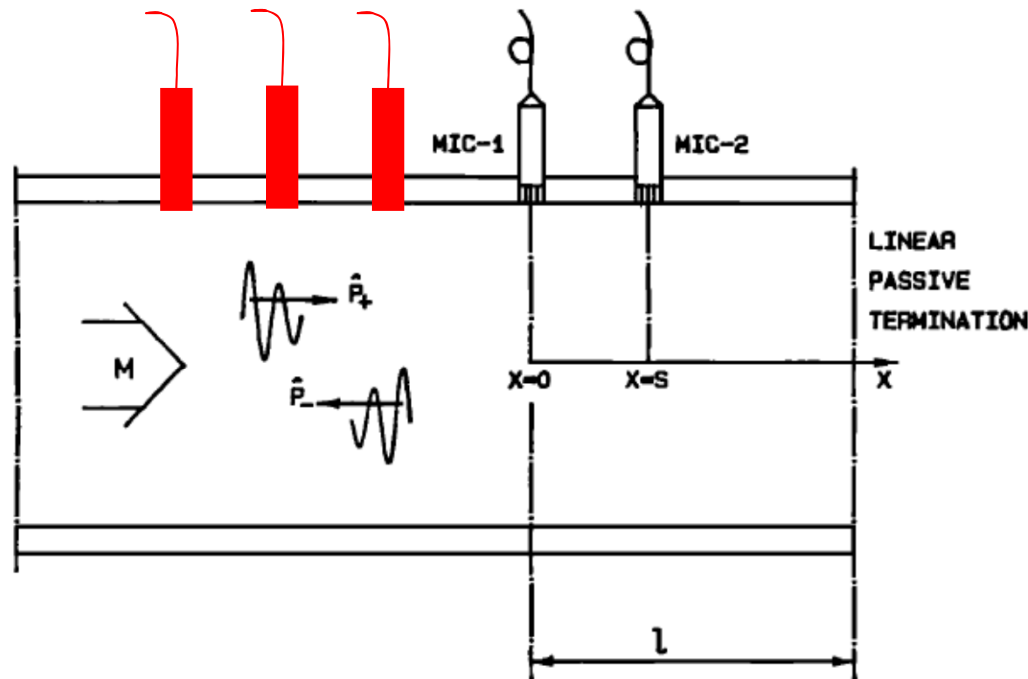
$$p_1(f) = p_+(f) + p_-(f)$$

$$p_2(f) = p_+(f) \cdot \exp(-jk_+s) + p_-(f) \cdot \exp(jk_-s)$$

⋮

$$p_n(f) = p_+(f) \cdot \exp(-jk_+s_n) + p_-(f) \cdot \exp(jk_-s_n)$$

## Over-determination



Add more  
microphones

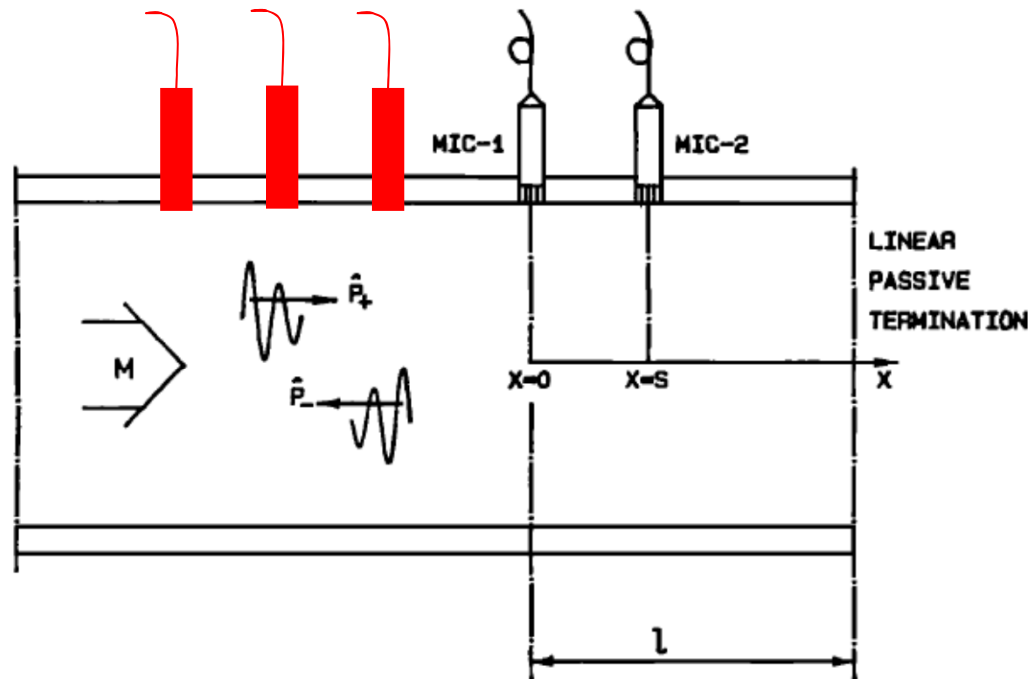
Can be used to  
reduce error

Accurate experimental two-port analysis of flow generated sound

Andreas Holmberg, Mats Åbom and Hans Bodén,  
Journal of Sound and Vibration 330 (2011) 6336–6354



Treat  $k_+s$  and  $k_-s$  as unknowns



Add more microphones

Solve nonlinear system of equations for  $k_+s$  and  $k_-s$

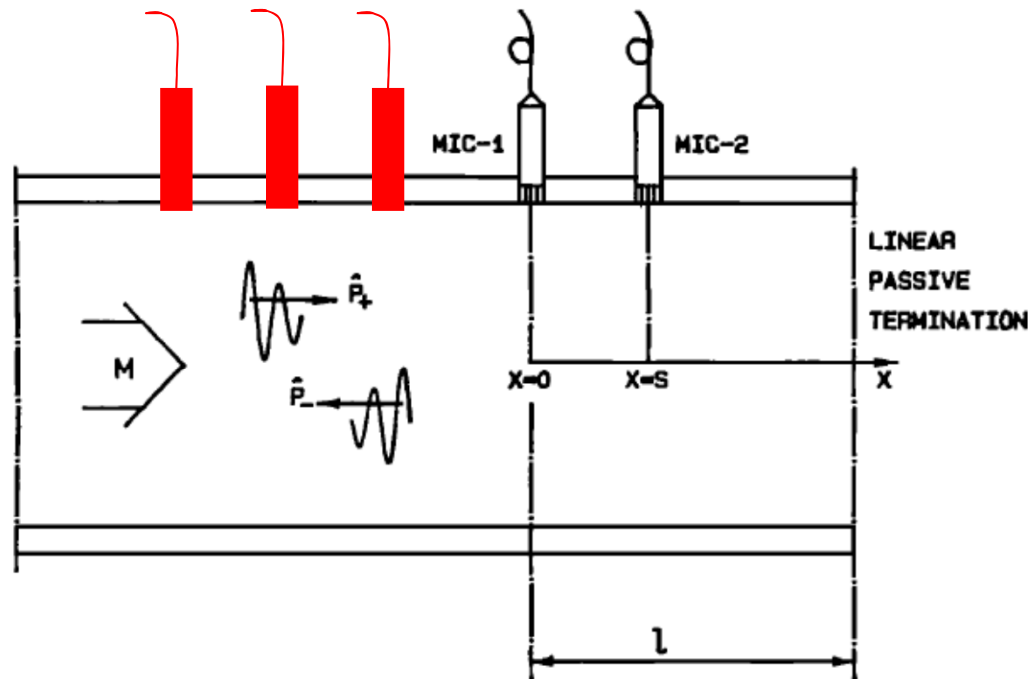
$$p_1(f) = p_+(f) + p_-(f)$$

$$p_2(f) = p_+(f) \cdot \exp(-jk_+s) + p_-(f) \cdot \exp(jk_-s)$$

⋮

$$p_n(f) = p_+(f) \cdot \exp(-jk_+s_n) + p_-(f) \cdot \exp(jk_-s_n)$$

Treat  $k_+$ s and  $k_-$ s as unknowns

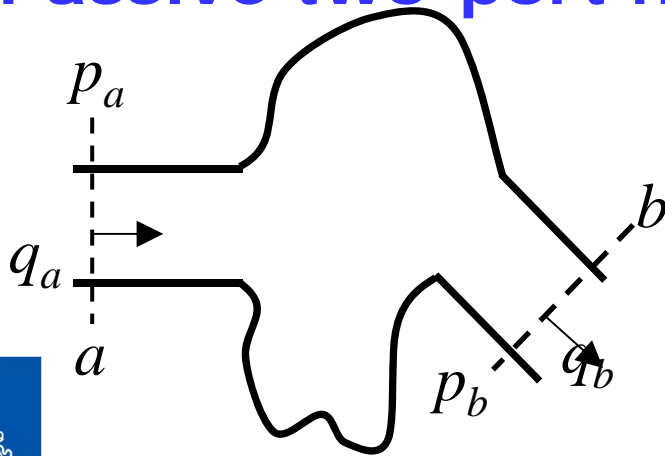


Add at least 4 microphones

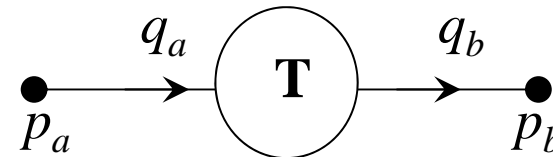
Solve nonlinear system of equations for  $k_+$ s and  $k_-$ s

S. Allam and M. Åbom, Investigation of damping and radiation using full plane wave decomposition in ducts. *Journal of Sound and Vibration* 292 (2006) 519-534. doi:10.1016/j.jsv.2005.08.016

# Passive two-port measurement techniques



Physical system

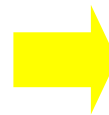


Equivalent circuit



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In the frequency domain a (linear) matrix relationship relates the states at  $a$  and  $b$ . A common choice of state variables is  $p$  and  $q$ .



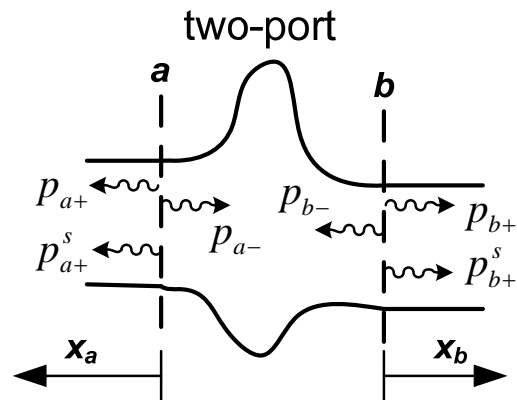
$$\begin{pmatrix} \hat{p}_a \\ \hat{q}_a \end{pmatrix} = \begin{pmatrix} T_{11} & T_{12} \\ T_{21} & T_{22} \end{pmatrix} \begin{pmatrix} \hat{p}_b \\ \hat{q}_b \end{pmatrix}$$

Mathematical model

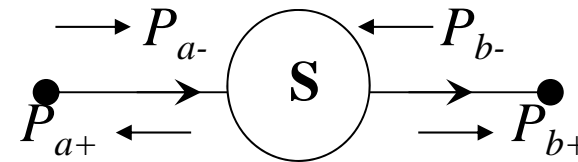
"Four pole"



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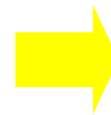


Physical system



Equivalent circuit

In the frequency domain a (linear) matrix relationship relates the states at  $a$  and  $b$ . Another common choice of state variables is  $p_+$  and  $p_-$ .



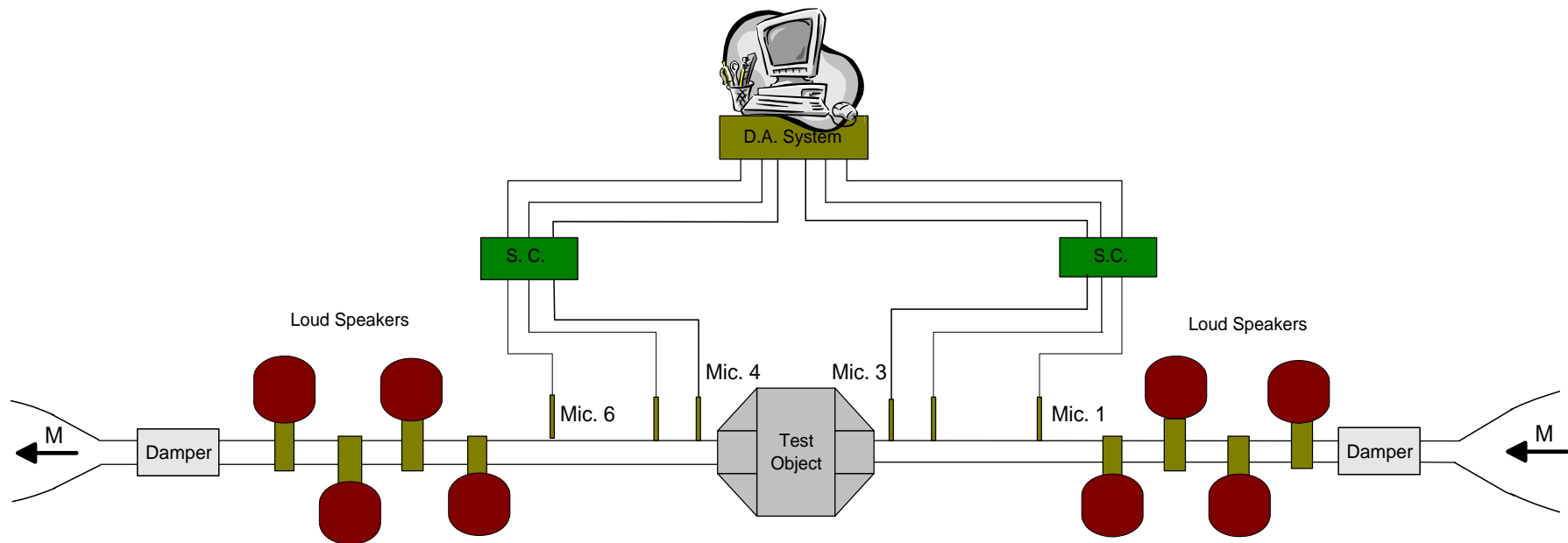
$$\begin{pmatrix} p_{a-} \\ p_{b-} \end{pmatrix} = \begin{pmatrix} S_{11} & S_{12} \\ S_{21} & S_{22} \end{pmatrix} \begin{pmatrix} p_{a+} \\ p_{b+} \end{pmatrix}$$

Mathematical model

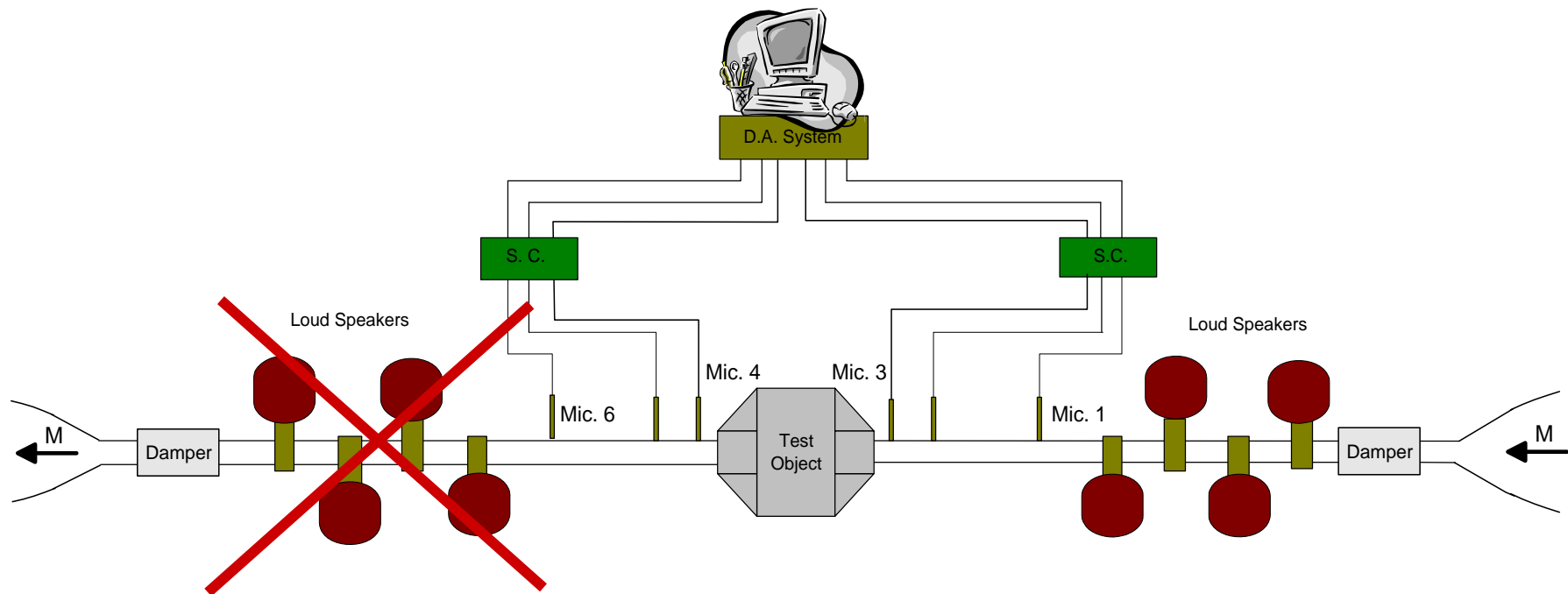
"Scattering matrix"



## Experimental determination of Two-port data using the Two-source technique



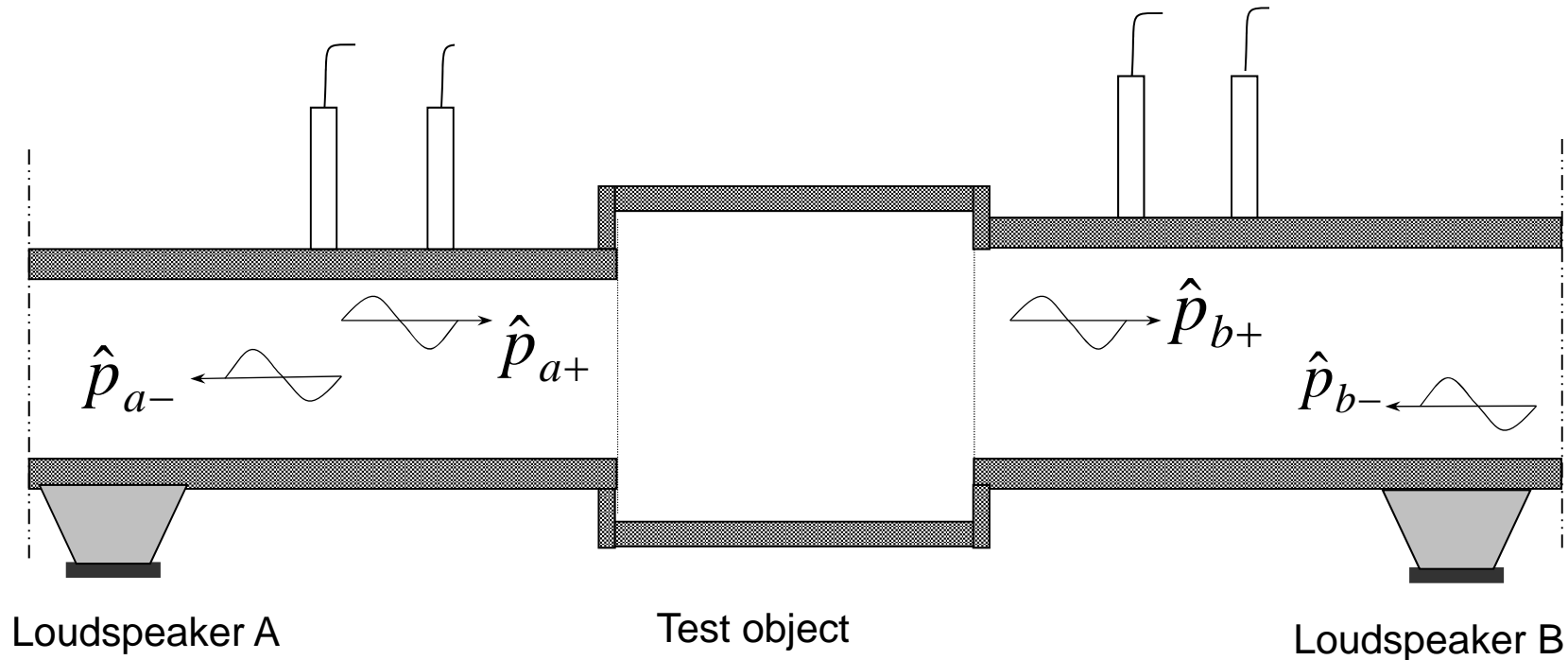
## Experimental determination of Two-port data using the Two-load technique



**Change acoustic load instead**

# Theoretical background

## Two-port measurements using the two source technique



$$\begin{pmatrix} p_a \\ q_a \end{pmatrix} = \begin{pmatrix} T_{aa} & T_{ab} \\ T_{ba} & T_{bb} \end{pmatrix} \begin{pmatrix} p_b \\ q_b \end{pmatrix} \quad \begin{aligned} p_a &= p_+ \exp(-i k_+^a L_a) + p_- \exp(i k_-^a L_a) \\ q_a &= \frac{A_a}{\rho c} \{ p_+ \exp(-i k_+^a L_a) - p_- \exp(i k_-^a L_a) \} \end{aligned}$$

## Theoretical background

$$\begin{pmatrix} p_a^1 & p_a^2 \\ q_a^1 & q_a^2 \end{pmatrix} = \begin{pmatrix} T_{aa} & T_{ab} \\ T_{ba} & T_{bb} \end{pmatrix} \begin{pmatrix} p_b^1 & p_b^2 \\ q_b^1 & q_b^2 \end{pmatrix}$$

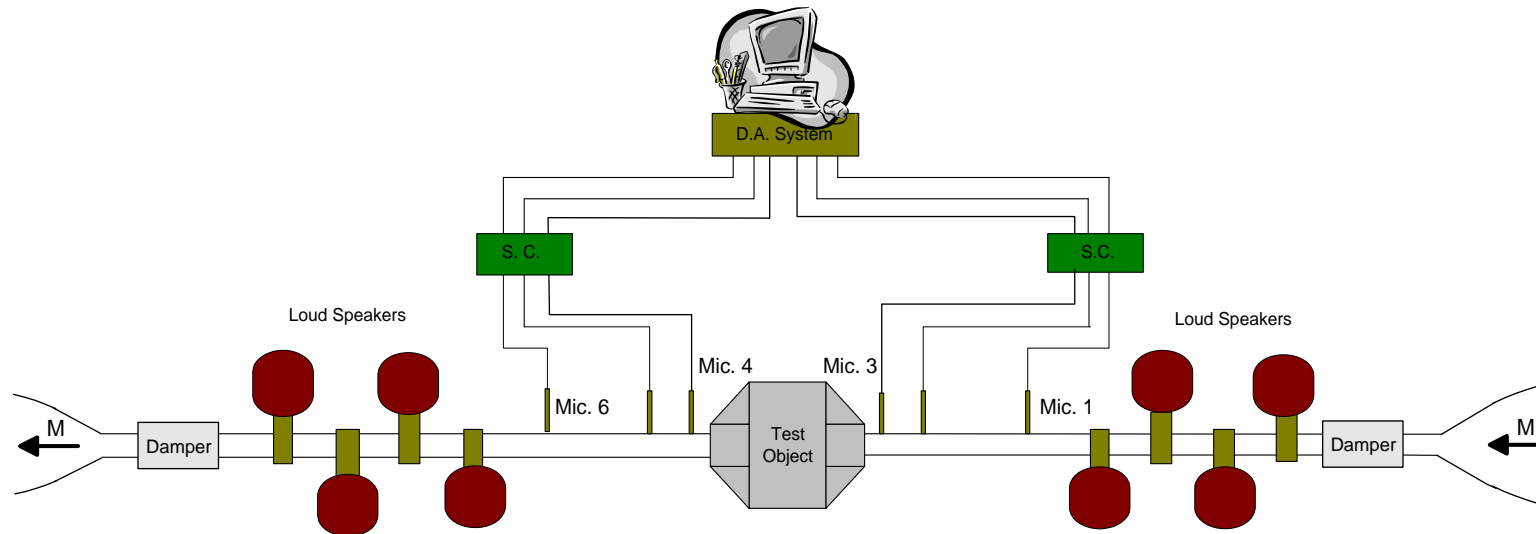
and the two-port matrix is determined from:

$$\begin{pmatrix} T_{aa} & T_{ab} \\ T_{ba} & T_{bb} \end{pmatrix} = \begin{pmatrix} p_a^1 & p_a^2 \\ q_a^1 & q_a^2 \end{pmatrix} \begin{pmatrix} p_b^1 & p_b^2 \\ q_b^1 & q_b^2 \end{pmatrix}^{-1}$$

if

$$\det \begin{pmatrix} p_b^1 & p_b^2 \\ q_b^1 & q_b^2 \end{pmatrix} \neq 0$$

## Typical test setup at MWL:



6-8 loudspeakers, two-source technique used,  
input signal available as reference

6-12 B&K ¼-inch microphones

Flow speed measured using Pitot tube and hotwire  
anemometer. Measurements made before and after the acoustic  
measurements.



# Flow Noise Supression

**Obtain initial good signal-to-noise ratio by:**

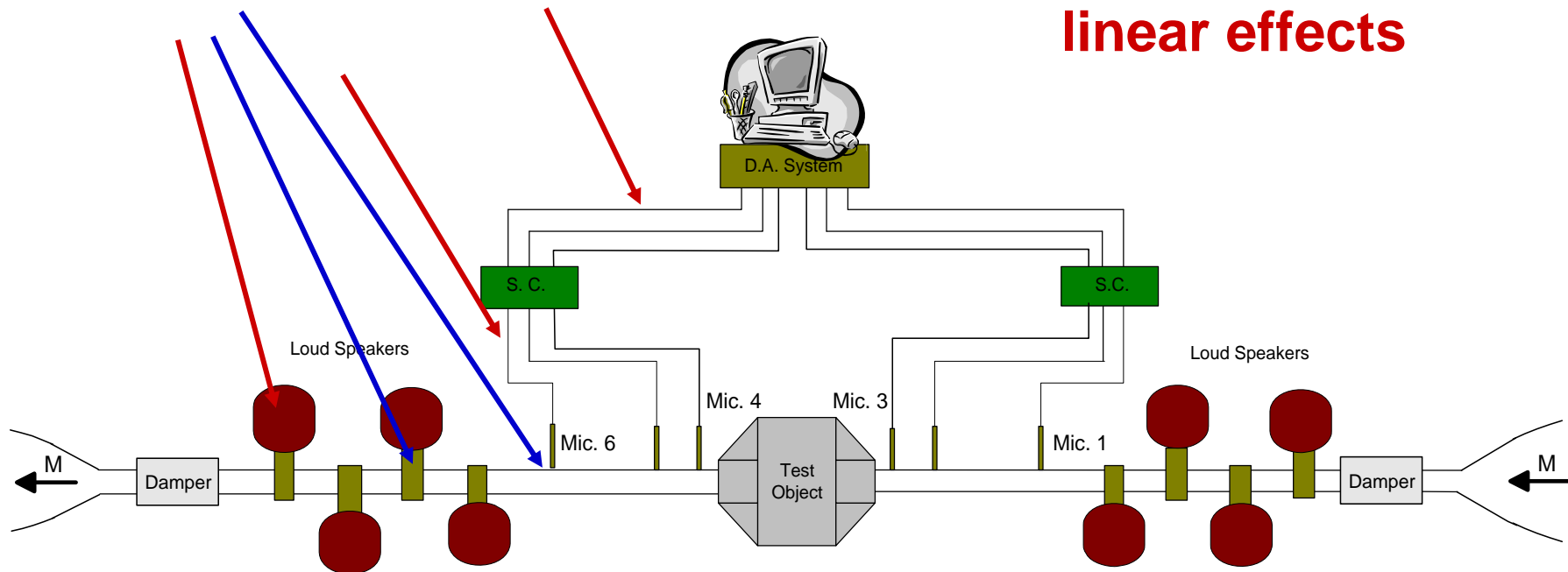
- Increasing level of input signal
- Using multiple loudspeakers
- Using high level loudspeakers
- Concentrating signal energy to narrow frequency bands

# Flow Noise Suppression

Use reference signal and "correlation" techniques:

Possible reference signal locations

Risk for non-linear effects



## Signal enhancement techniques

- 1) Frequency domain averaging (**FDA**). Welch's technique.
- 2) Synchronised time domain averaging (**STDA**). Requires deterministic signal + reference (trig) signal.
- 3) Cross-spectrum based frequency domain averaging (**CSFDA**). Requires noise free reference.

Auto-spectrum estimation

$$\hat{G}_{xx}(\omega) = \frac{\hat{G}_{rx}^*(\omega) \hat{G}_{rx}(\omega)}{\hat{G}_{rr}(\omega)}$$

Averaging

$$\hat{G}_{\gamma x}(\omega) = \frac{1}{N} \sum_{i=1}^N G_{\gamma x}^i(\omega)$$

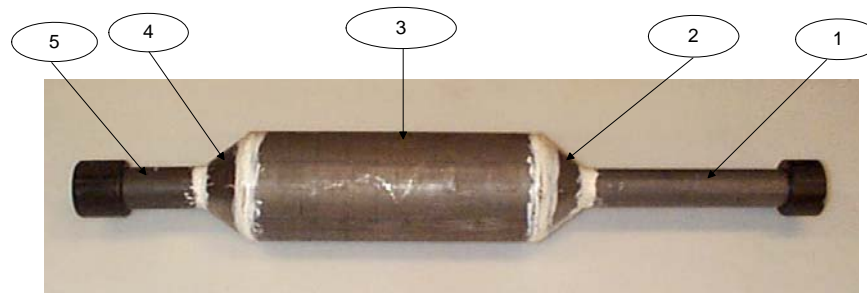
An improvement in SNR by a factor of  $N$  can be expected or  $10 \cdot \text{Log}(N)$  dB

# Test objects for two-port measurements

## Three configurations tested

1) Straight hard-walled pipe. Theoretical solution available.

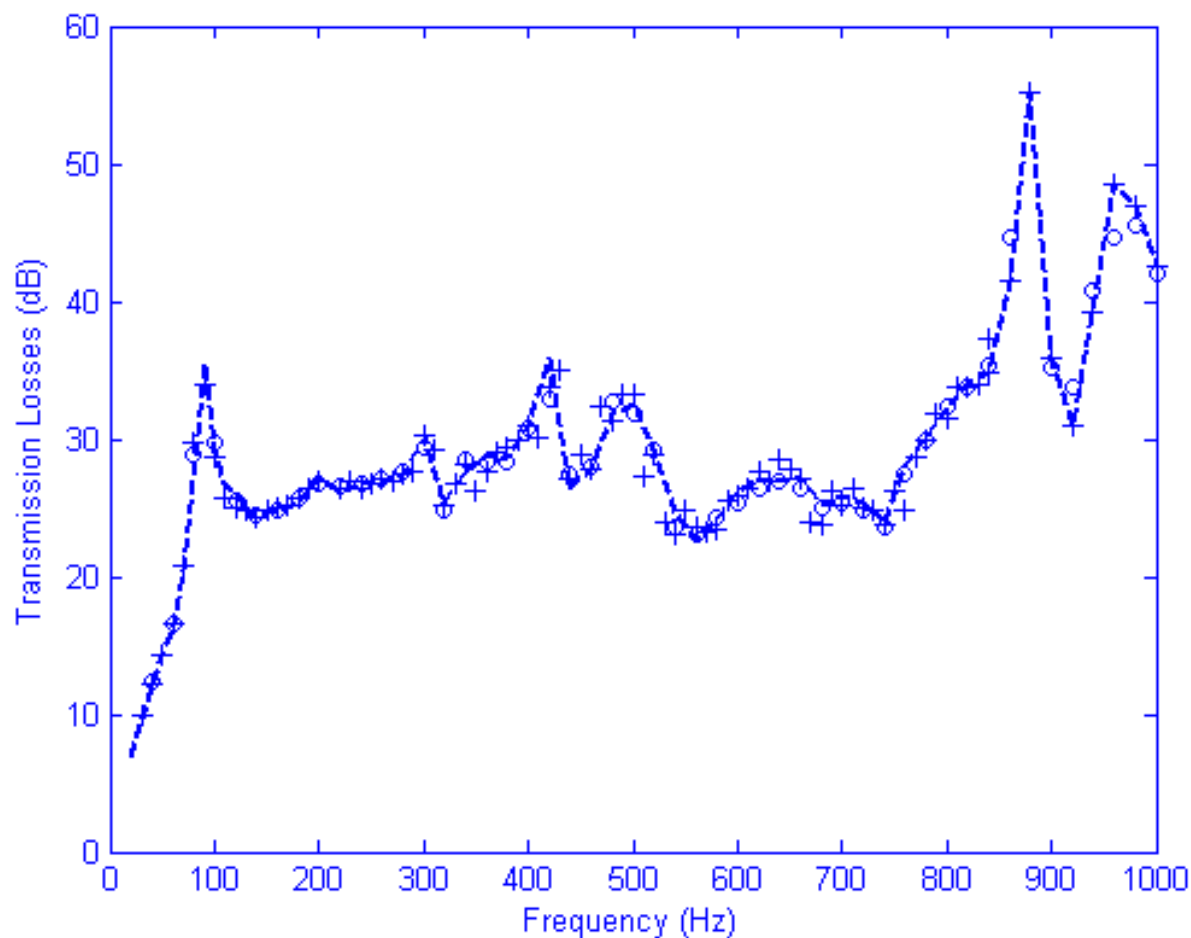
2) Expansion chamber



3) Automotive muffler



# Commercial muffler



Transmission loss at  
 $M=0.26$ .

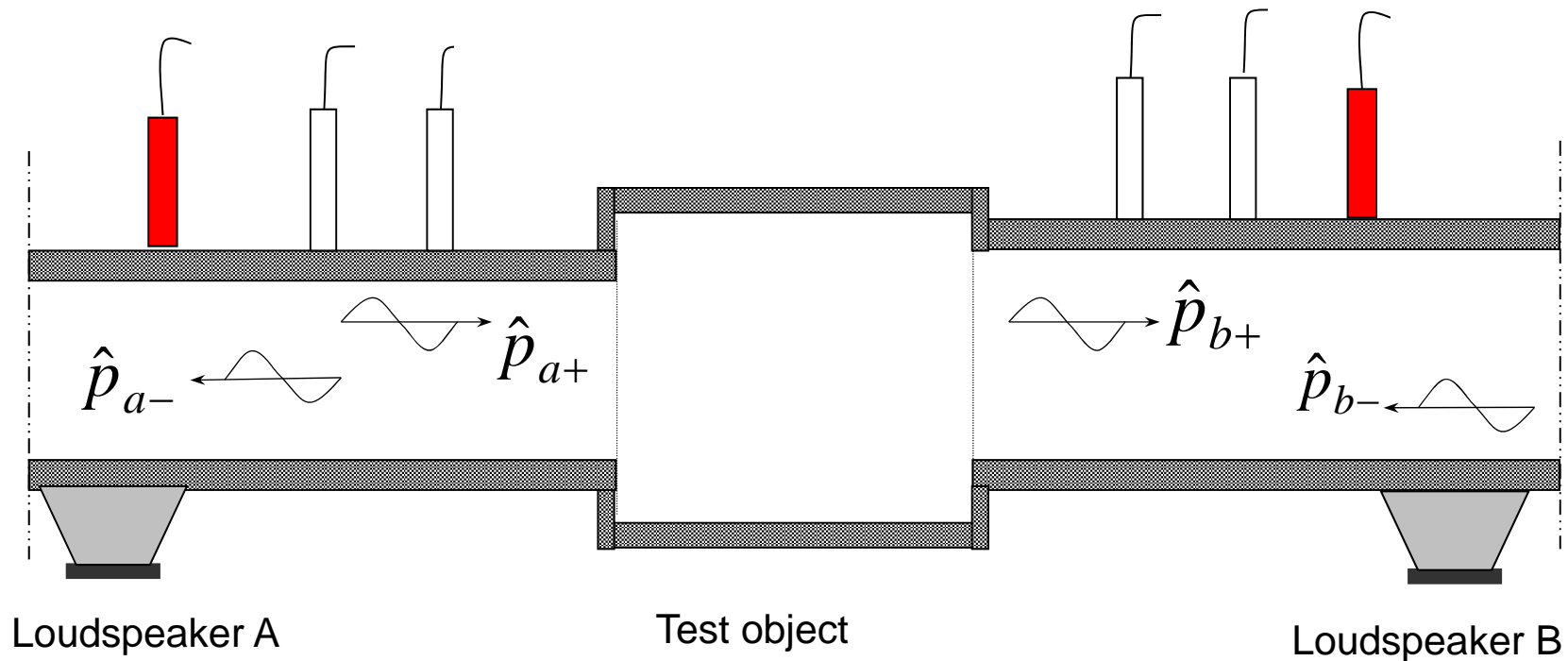
----, random excitation  
10000 averages (CSFDA)

++++, sawtooth excitation  
10000 averages (STDA)

oooo, stepped sine  
excitation 400 averages  
(CSFDA)



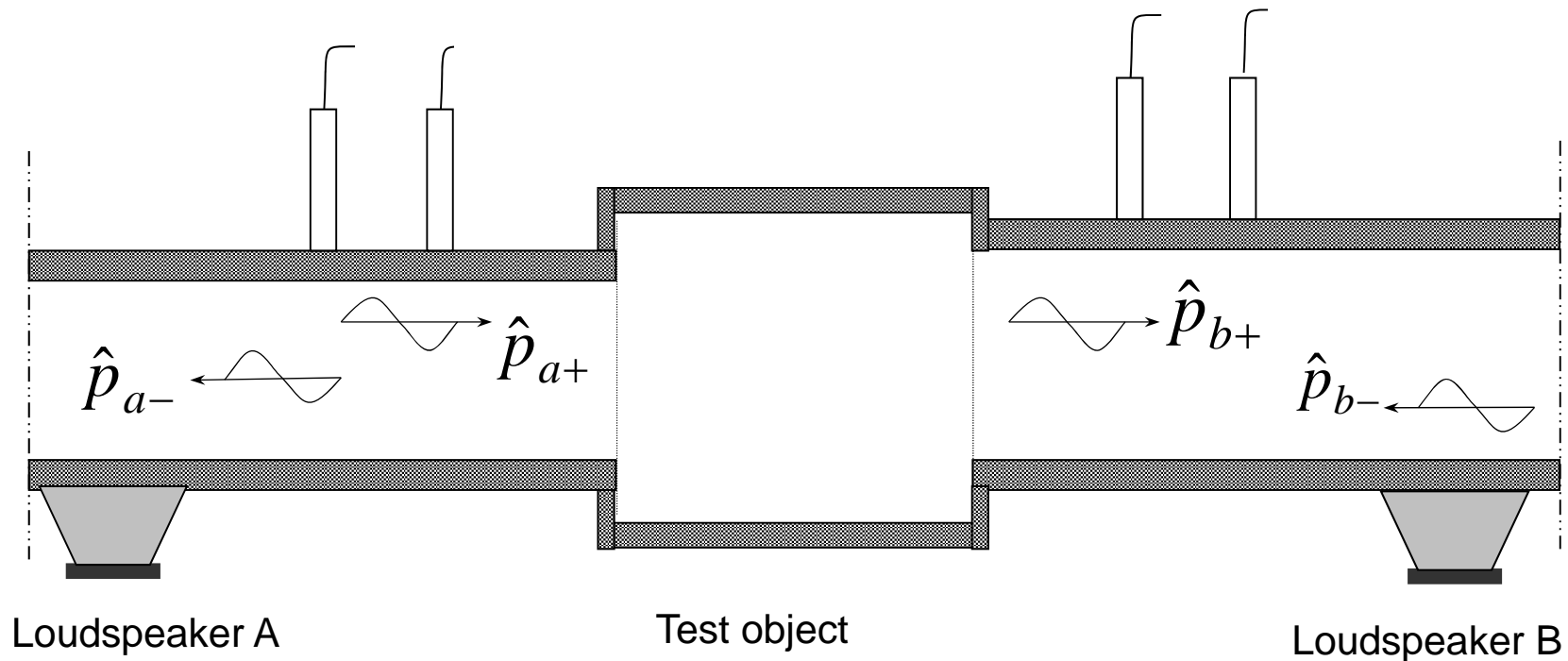
## Two-port measurements with over-determination



$$\begin{pmatrix} p_a \\ q_a \end{pmatrix} = \begin{pmatrix} T_{aa} & T_{ab} \\ T_{ba} & T_{bb} \end{pmatrix} \begin{pmatrix} p_b \\ q_b \end{pmatrix}$$

**Add additional microphones**

## Two-port measurements with over-determination



$$\begin{pmatrix} p_a \\ q_a \end{pmatrix} = \begin{pmatrix} T_{aa} & T_{ab} \\ T_{ba} & T_{bb} \end{pmatrix} \begin{pmatrix} p_b \\ q_b \end{pmatrix}$$

**Use: A, B, A+B, A-B, B-A.....**  
**Does not give additional independent equations**

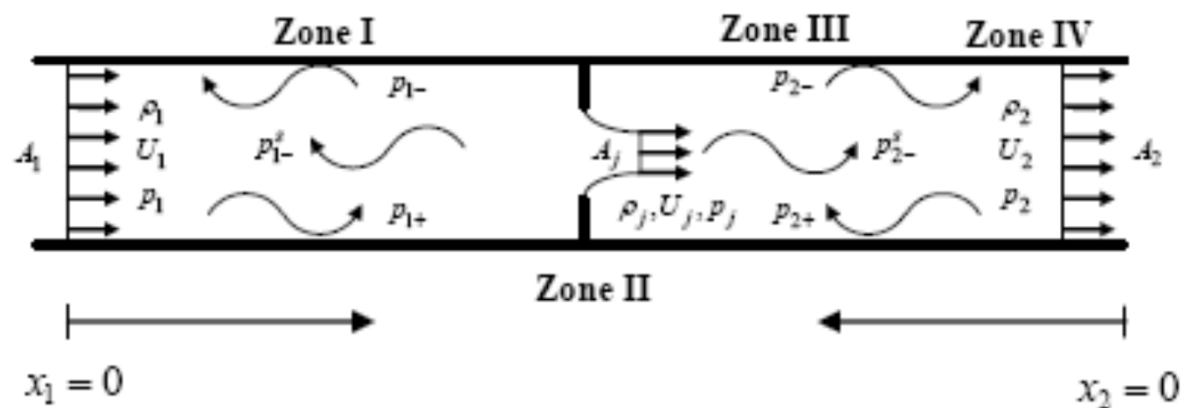
## Results and discussion

Measurements made for single diaphragm orifice

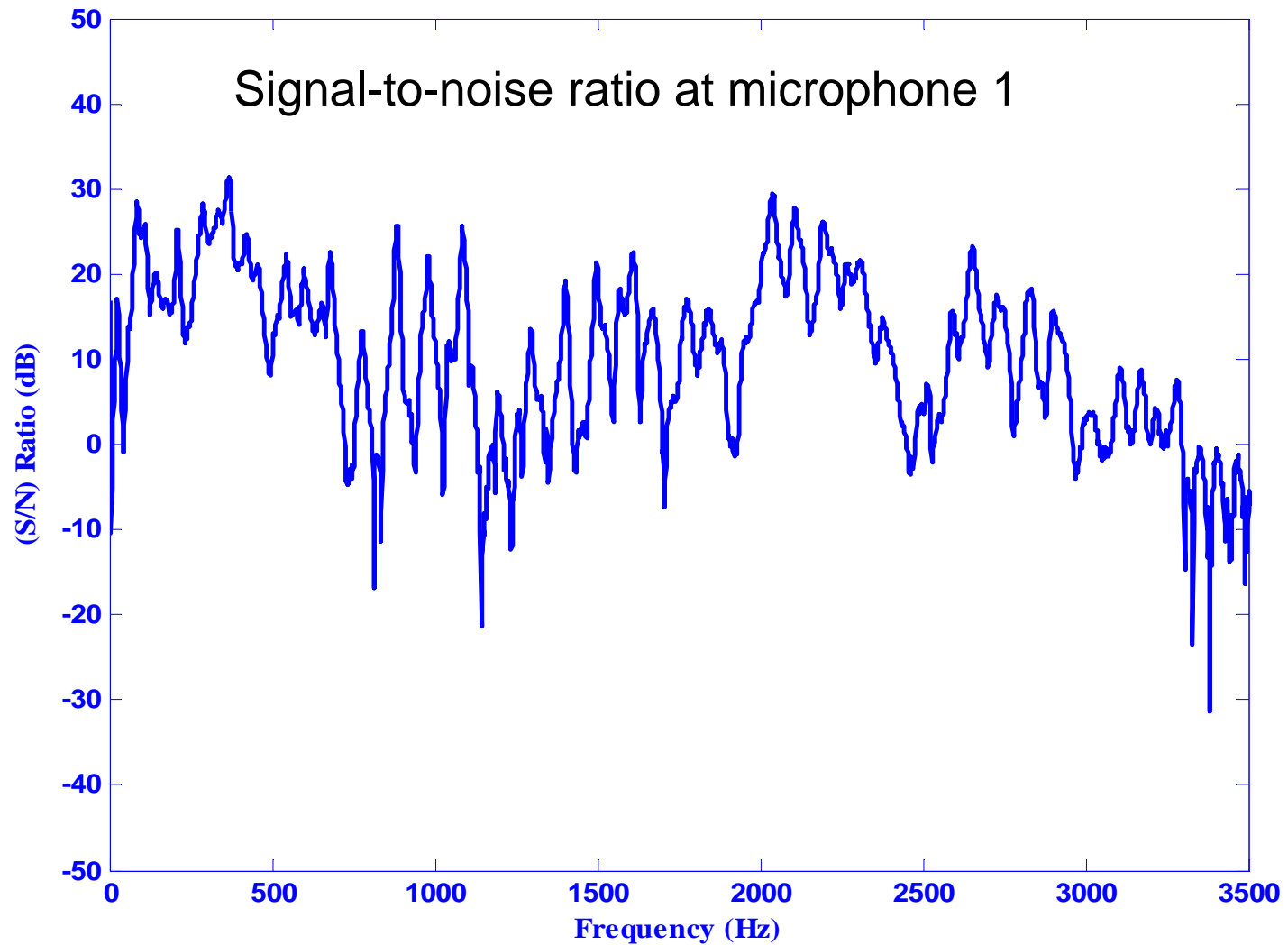
5 different test cases

Comparison to FEM prediction (FEMLAB)

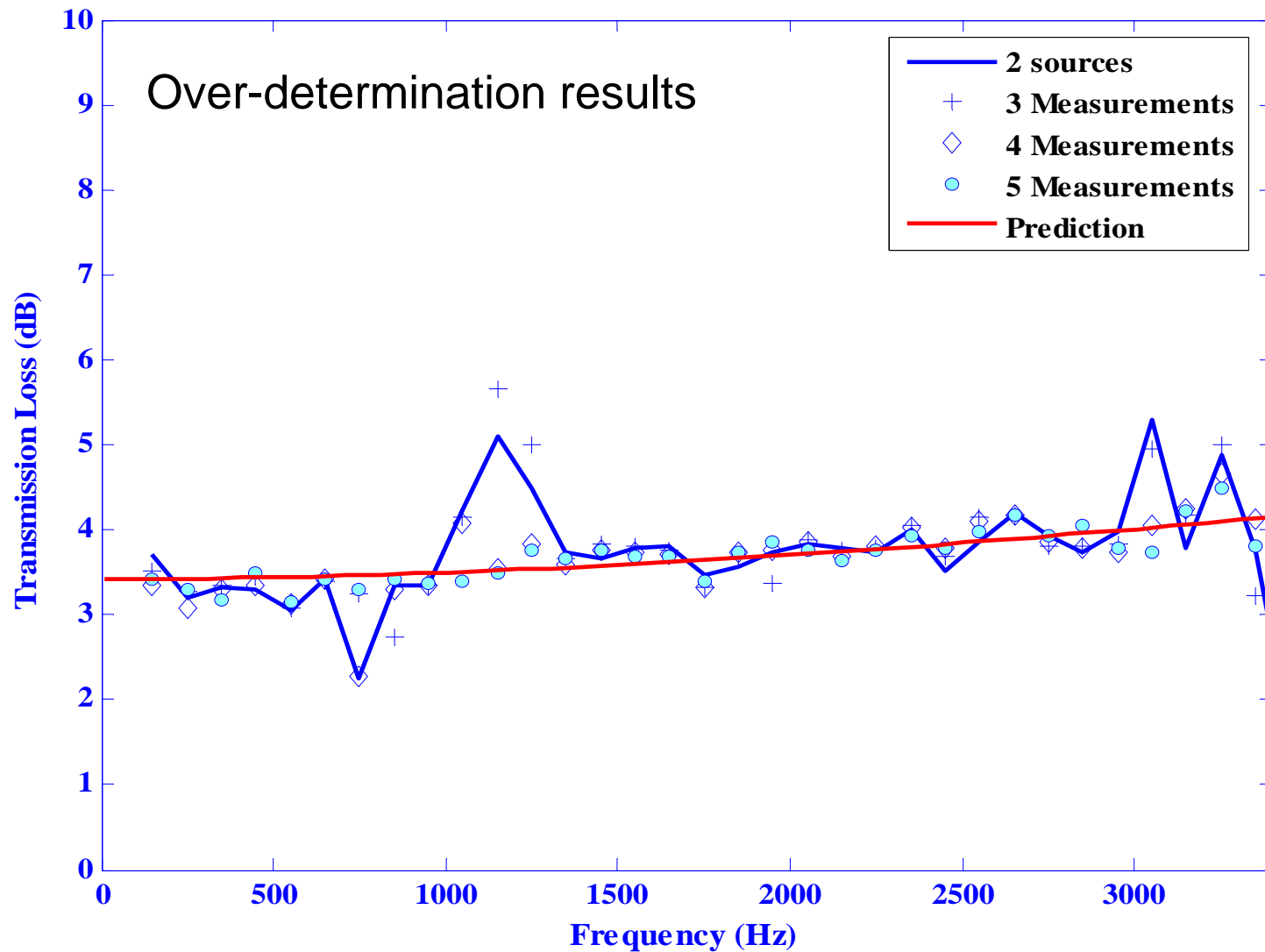
Low signal-to-noise ratio



# Results and discussion



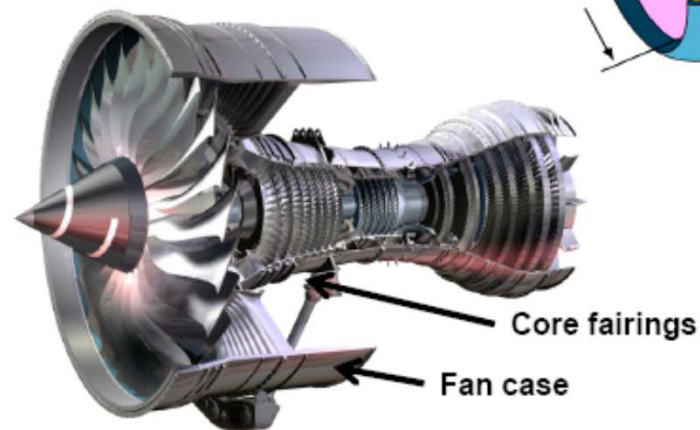
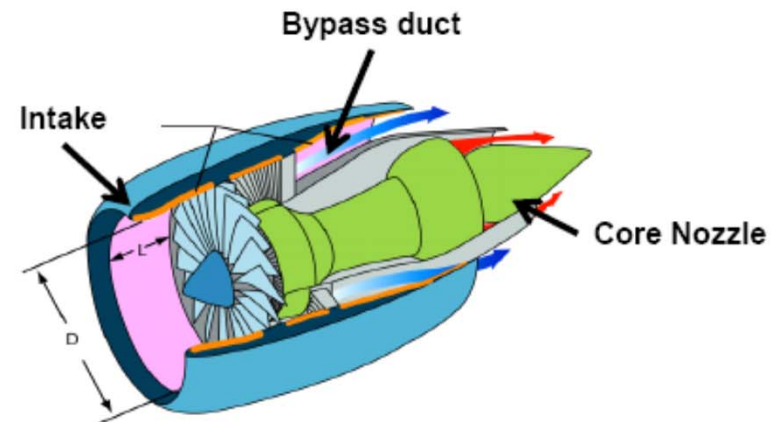
## Results and discussion



# Aero-Engine Liners



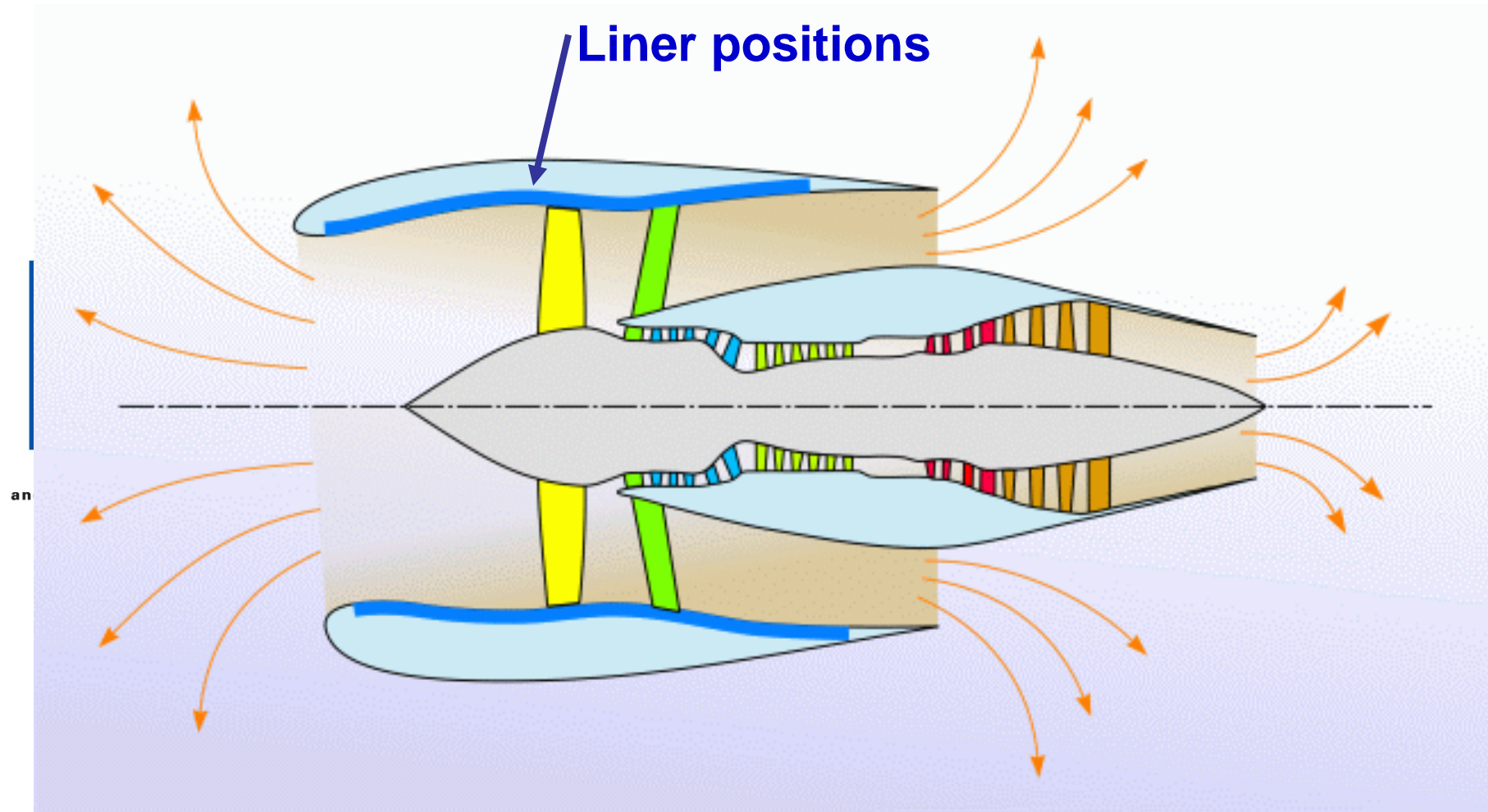
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Rolls-Royce



## Aircraft Engine Liners

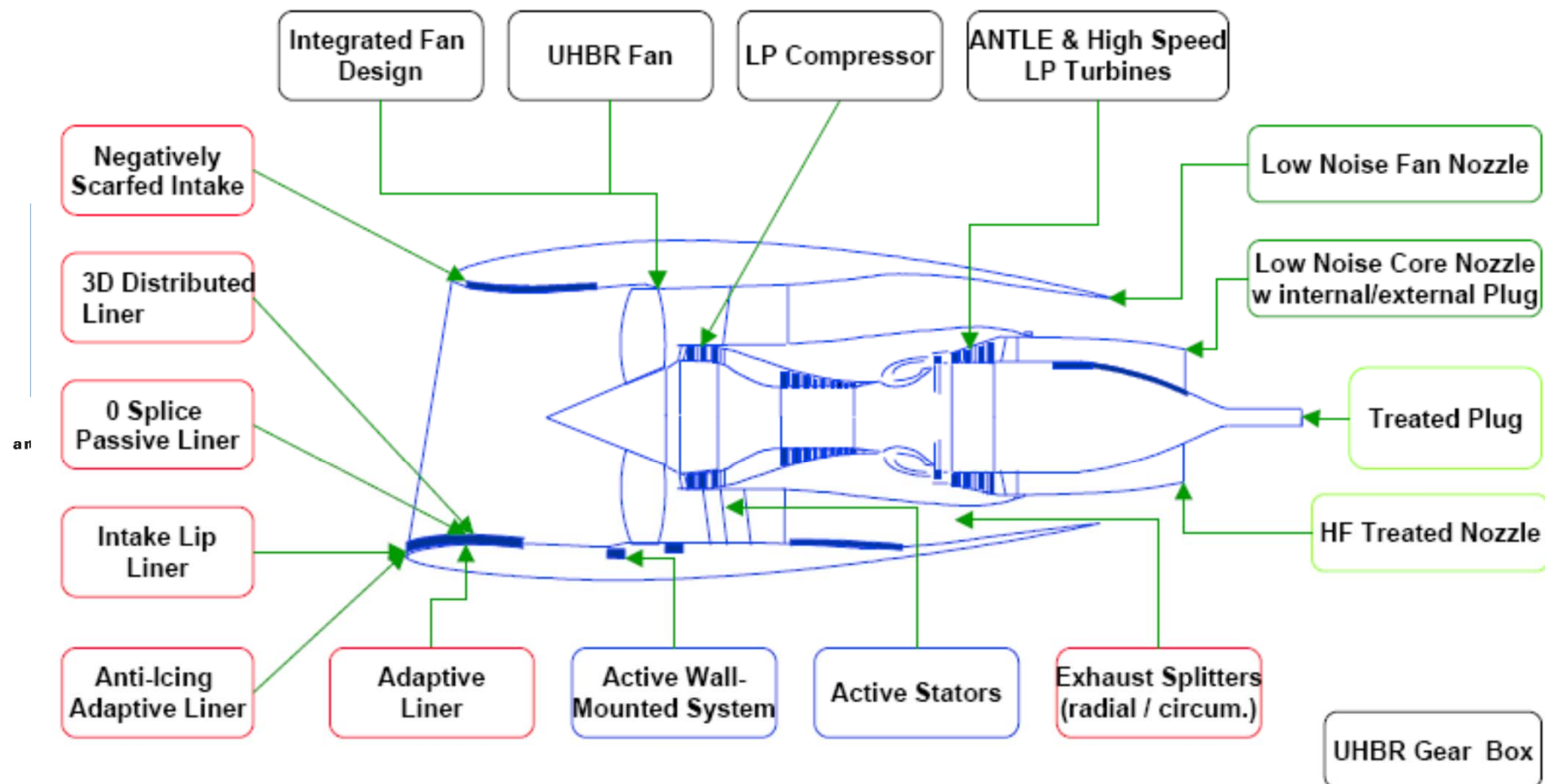




*SILENCE(R)*



## *How to reach Objectives*



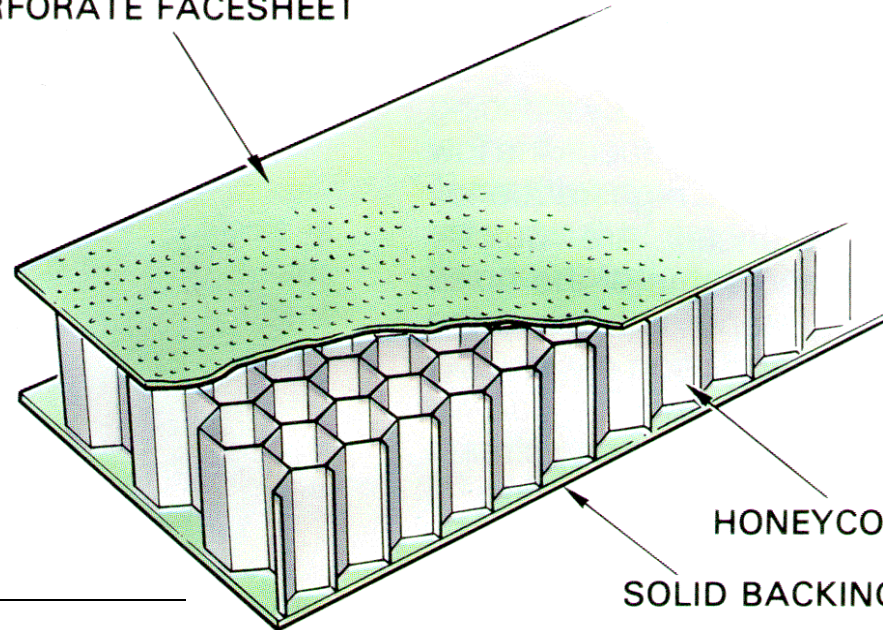
Background

# Aircraft Engine Liner



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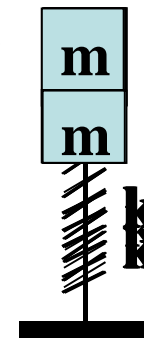
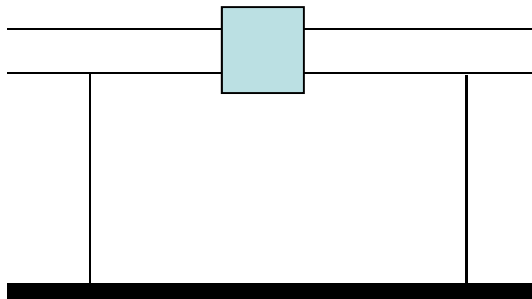
PERFORATE FACESHEET



HONEYCOMB SUPPORT

SOLID BACKING SHEET

## Helmholtz Resonator

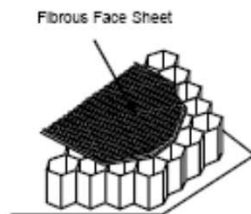


## Liner Types in common use

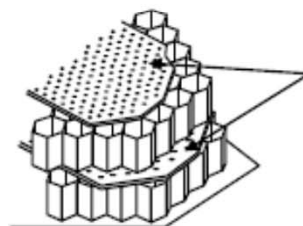


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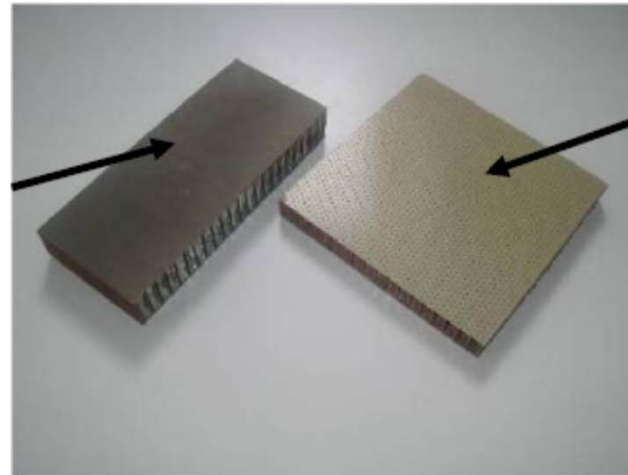
*Single Layer Linear*  
Wire mesh on perforate  
sheet + honeycomb +  
hardwall backing sheet



*Double layer liners*

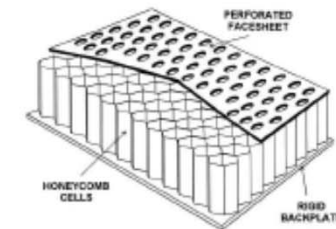


Perforate or  
Linear  
Face and  
Septum



*Single Layer Perforate*

Perforate sheet + honeycomb +  
hardwall backing sheet



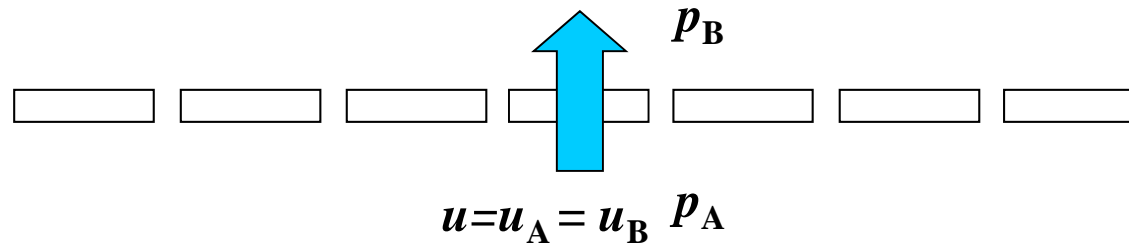
**Perforated sheets**

- Punched Aluminium
- Mechanically Drilled Carbon Fibre Composites
- Laser Drilled
- Injection Moulded
- Microperforates



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## Normalised Acoustic Impedance



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$$Z = \frac{\Delta p}{\rho_0 c u} = \frac{(1 - \sigma^2 C_D^2)}{2c C_D^2} \cdot |u| + jkl$$

**This neglects linear (viscous) losses, flow effects etc.**

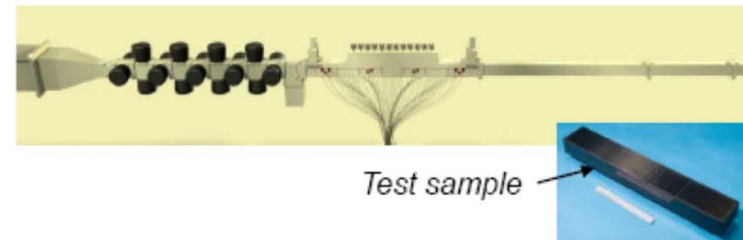


## Liner Impedance Measurements

DC Flow Resistance



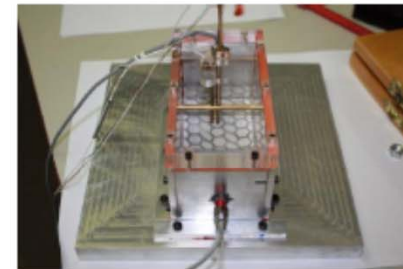
NASA Grazing Flow Impedance Tube



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Portable Acoustic Liner Meter



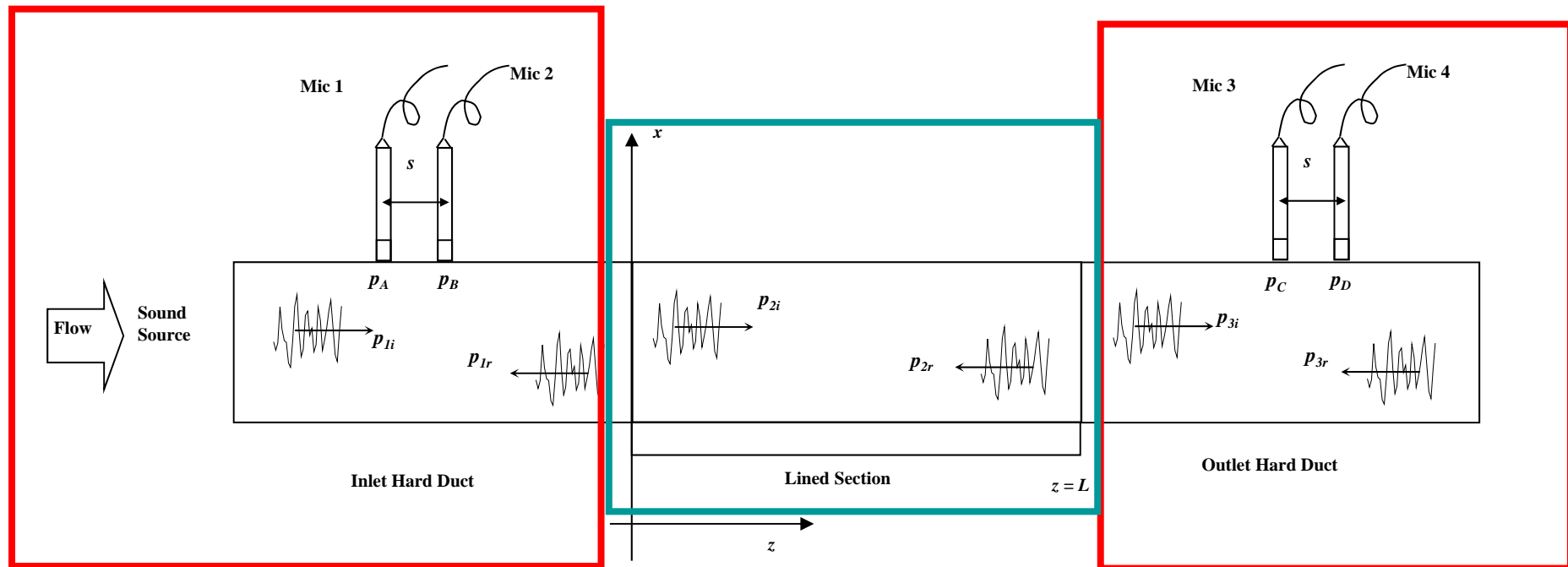
NLR In-Situ Impedance Measurements



Rolls-Royce



# Grazing flow impedance “eduction” – Inverse determination of impedance boundary condition



Model for sound propagation in lined section

Plane wave decomposition

# Models for perforate impedance

Elnady, T.,

Modelling and characterization of perforates in lined ducts and mufflers, *PhD Thesis*, KTH, Stockholm, Sweden, 2004.

## Normalised resistance

$$\theta = \operatorname{Re} \left( \frac{jk}{\sigma C_D} \left[ \frac{t}{F(\mu')} + \frac{\delta_{re}}{F(\mu)} f_{\text{int}} \right] \right) + \frac{1}{\sigma} \left[ 1 - \frac{2J_1(kd)}{kd} \right] + \left( \frac{1-\sigma^2}{\sigma^2 C_D^2} \right) \cdot \frac{1}{2c} |v_n| + \frac{0.5}{\sigma} \cdot M$$

## Normalised reactance

Nonlinear terms

Flow terms

$$X = \operatorname{Im} \left( \frac{jk}{\sigma C_D} \left[ \frac{t}{F(\mu')} + \frac{0.5d}{F(\mu)} f_{\text{int}} \right] \right) - \left( \frac{1-\sigma^2}{\sigma^2 C_D^2} \right) \cdot \frac{1}{2c} \cdot \frac{|v_n|}{3} - \frac{0.3}{\sigma} \cdot M$$

$$K = \sqrt{-\frac{j\omega}{\nu}}$$

$$\delta_{re} = 0.2d + 200d^2 + 16000d^3$$

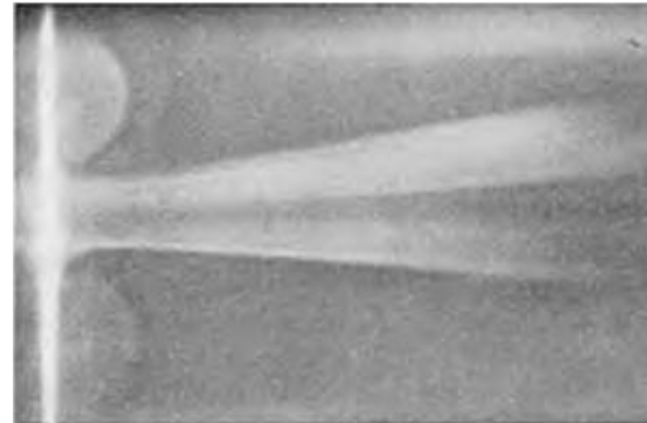
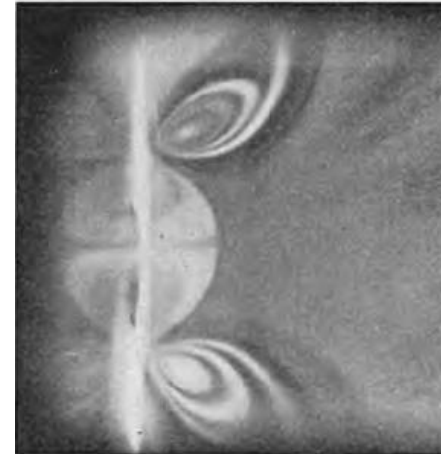
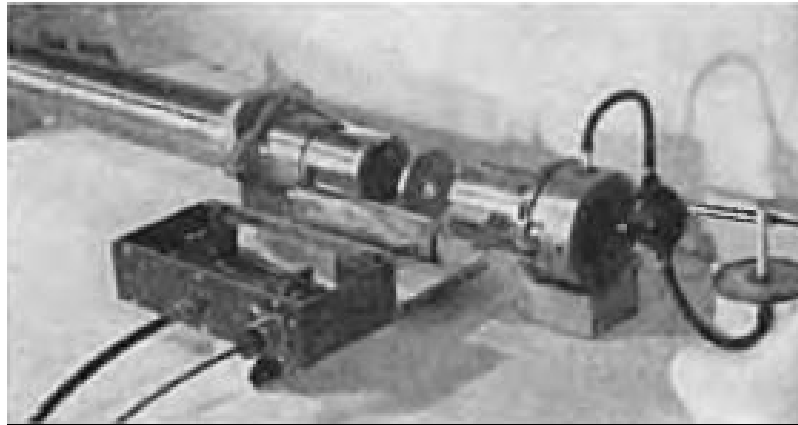
$$F(\mu) = 1 - \frac{4J_1(Kd/2)}{Kd \cdot J_0(Kd/2)}$$

$$f_{\text{int}} = 1 - 1.47\sqrt{\sigma} + 0.47\sqrt{\sigma^3}$$

## The acoustic properties can become non-linear at high acoustic excitation levels



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Ingård and Labate JASA, 1950

## The acoustic properties can become non-linear at high acoustic excitation levels



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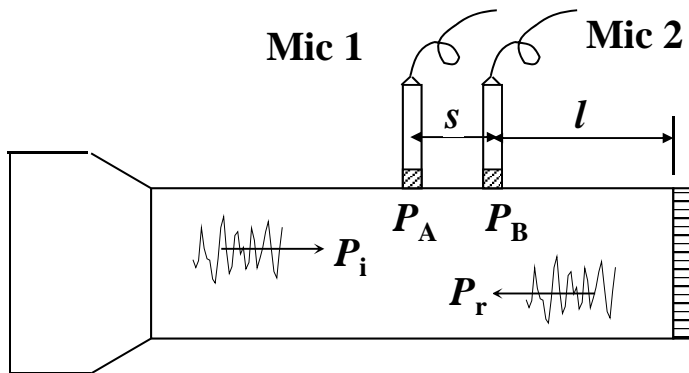


Fig. 28. Vortex shedding at a 90° slit.

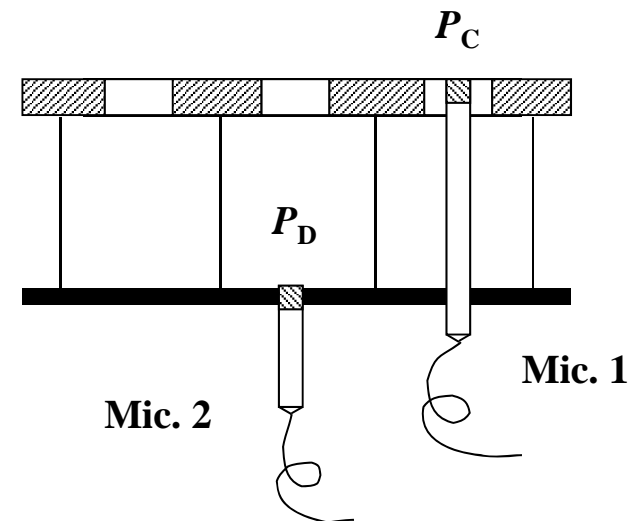
Tam et. al. JSV 284, 205

# Normal Incidence Impedance Measurement Techniques

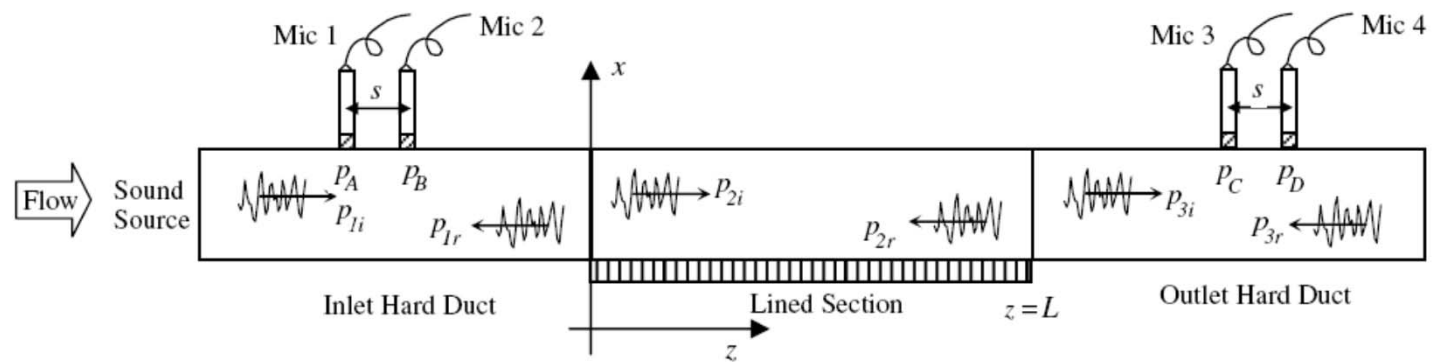
In-duct, two microphone technique:

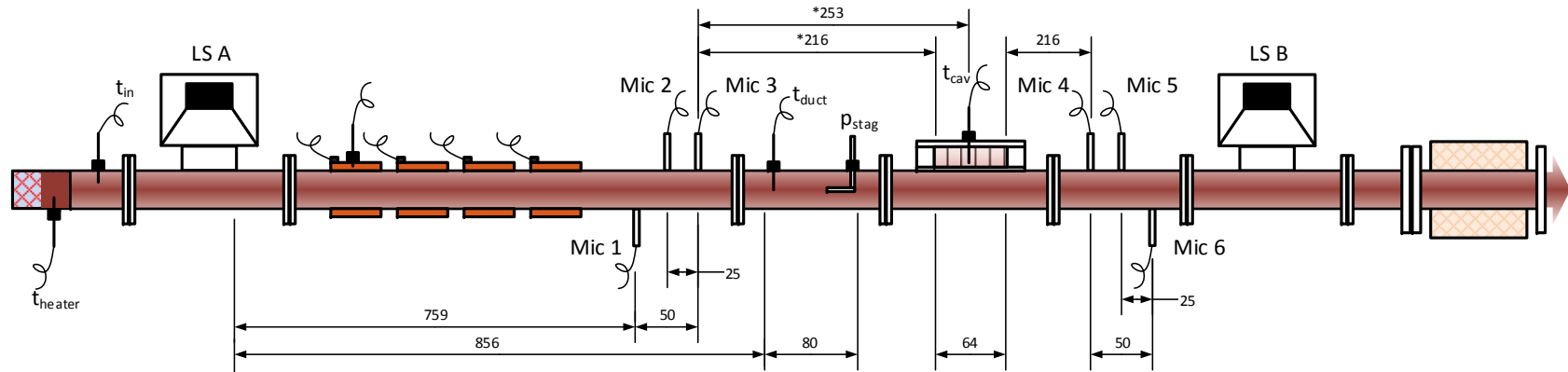


In-situ two-microphone technique:



# Grazing Flow Impedance Measurements

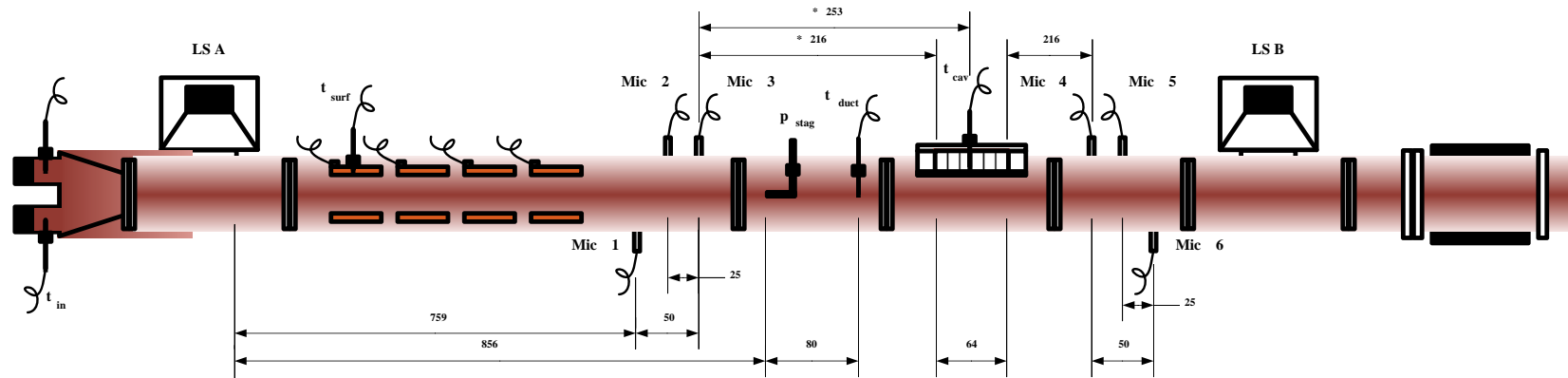




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## High flow speed test rig

## High temperature test rig



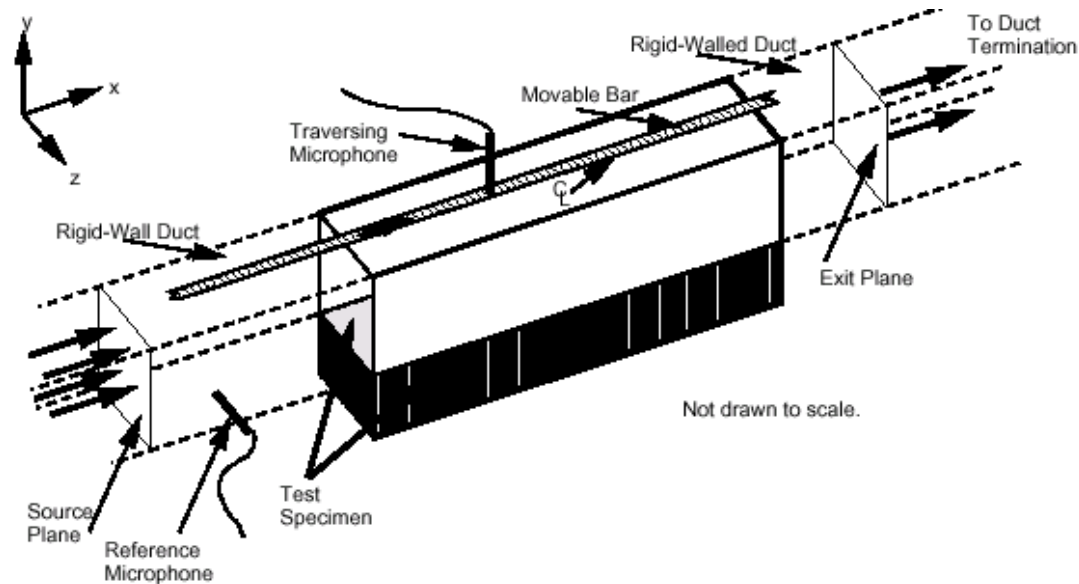


# Impedance Eduction

- W. Watson, M. Jones, T. Parrott, S. Tanner  
(NASA Langley Research Center)



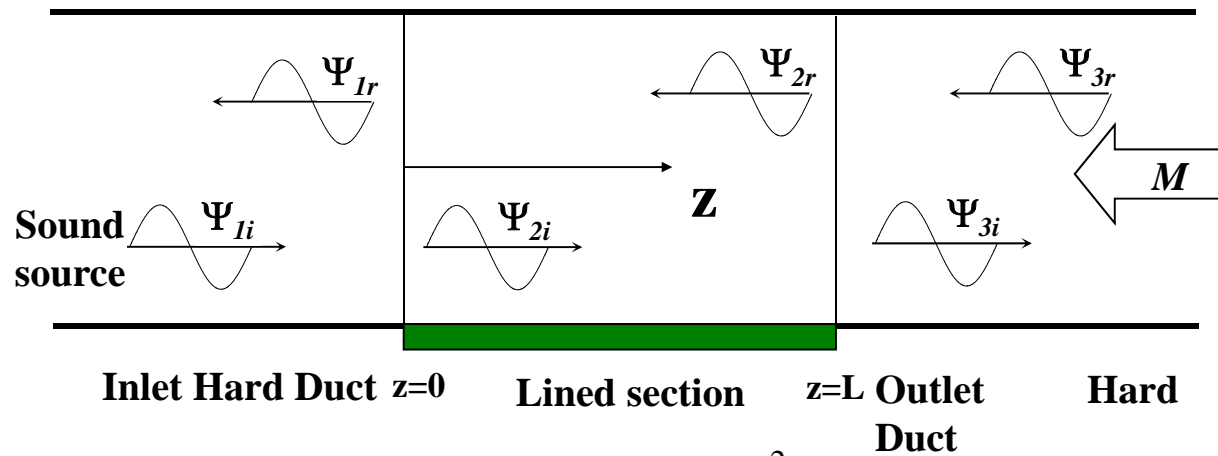
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# Computation Domain



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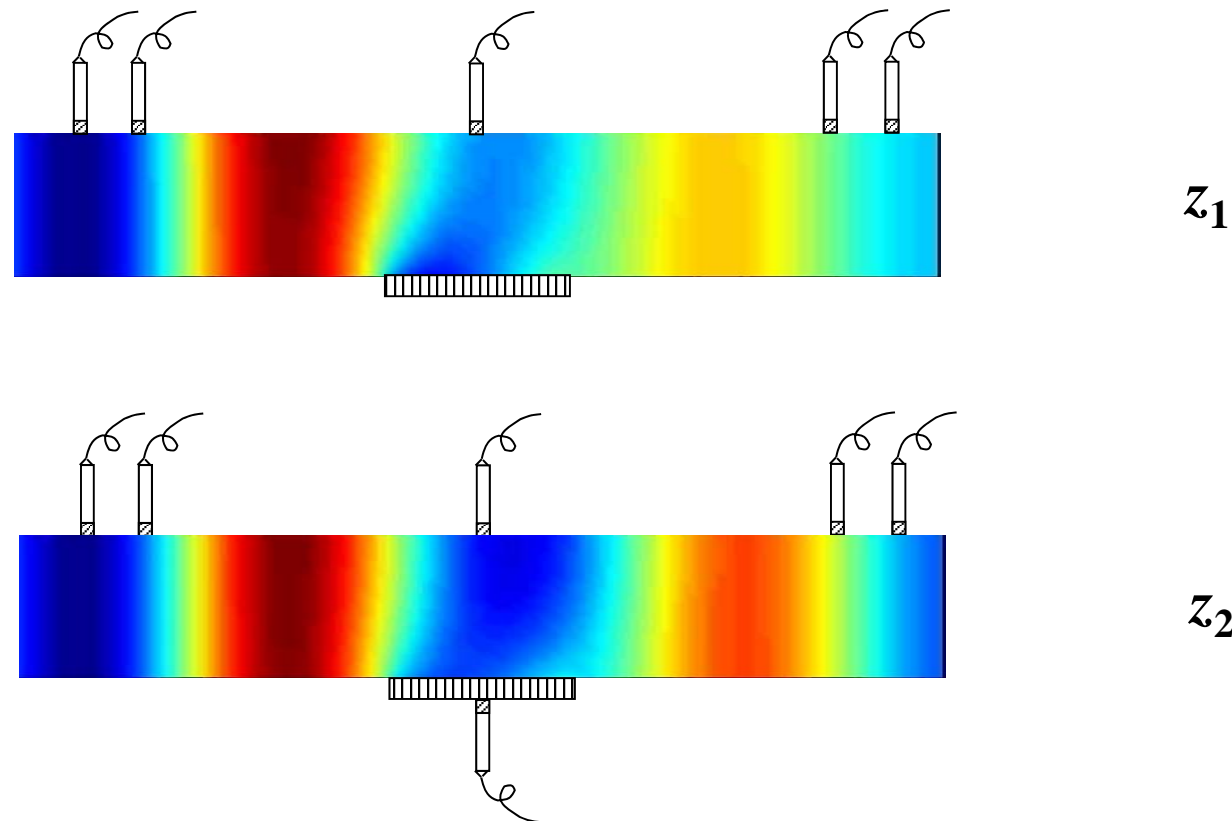
$$\nabla^2 \Psi_2 - \left( jk + M \frac{\partial}{\partial z} \right)^2 \Psi_2 = 0$$

$$\Psi_2(x, y, z) = \sum_{q=1}^Q b^{(q)} \cdot \psi_2^{(q)}(x, y) \cdot e^{-jk_z^{(q)} \cdot z}$$

## Experimental determination of grazing flow impedance



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# Education procedures

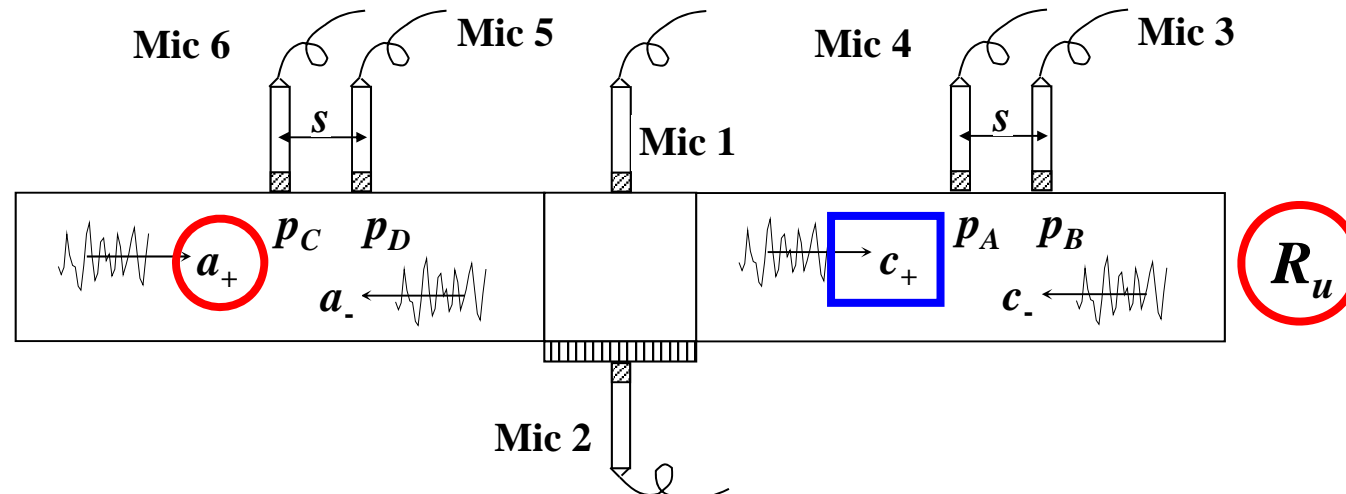
## Convected Hemholtz Equation Mode matching

## Straight Forward Method

## Pridmore Brown Equation



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## **Convected Hemiholtz Equation Mode matching**

**Analytical solution for sound field in lined section  
with known (guessed) impedance**

**Mode matching between lined and hard sections**

**Optimization, minimizing the difference between  
measured and calculated pressures at the  
microphone positions.**

## **KTH, Straight forward method**

**Assume only one propagating mode in the lined section**

**Measure the scattering matrix of the lined section**

**Calculate transverse wave numbers from the scattering  
matrix**

**Calculate the axial wave number assuming no reflection**

**Calculate the impedance**

**Comparison of impedance eduction results using different methods and test rigs. L. Zhou, H. Bodén, C. Lahiri, F. Bake, L. Enghardt and T. Elnady. AIAA/CEAS Aeroacoustics Conference, Atlanta 2014.**



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## **DLR, LINUS method**

**Measure the scattering matrix of the lined section**

**Calculate transverse wave numbers from the scattering matrix**

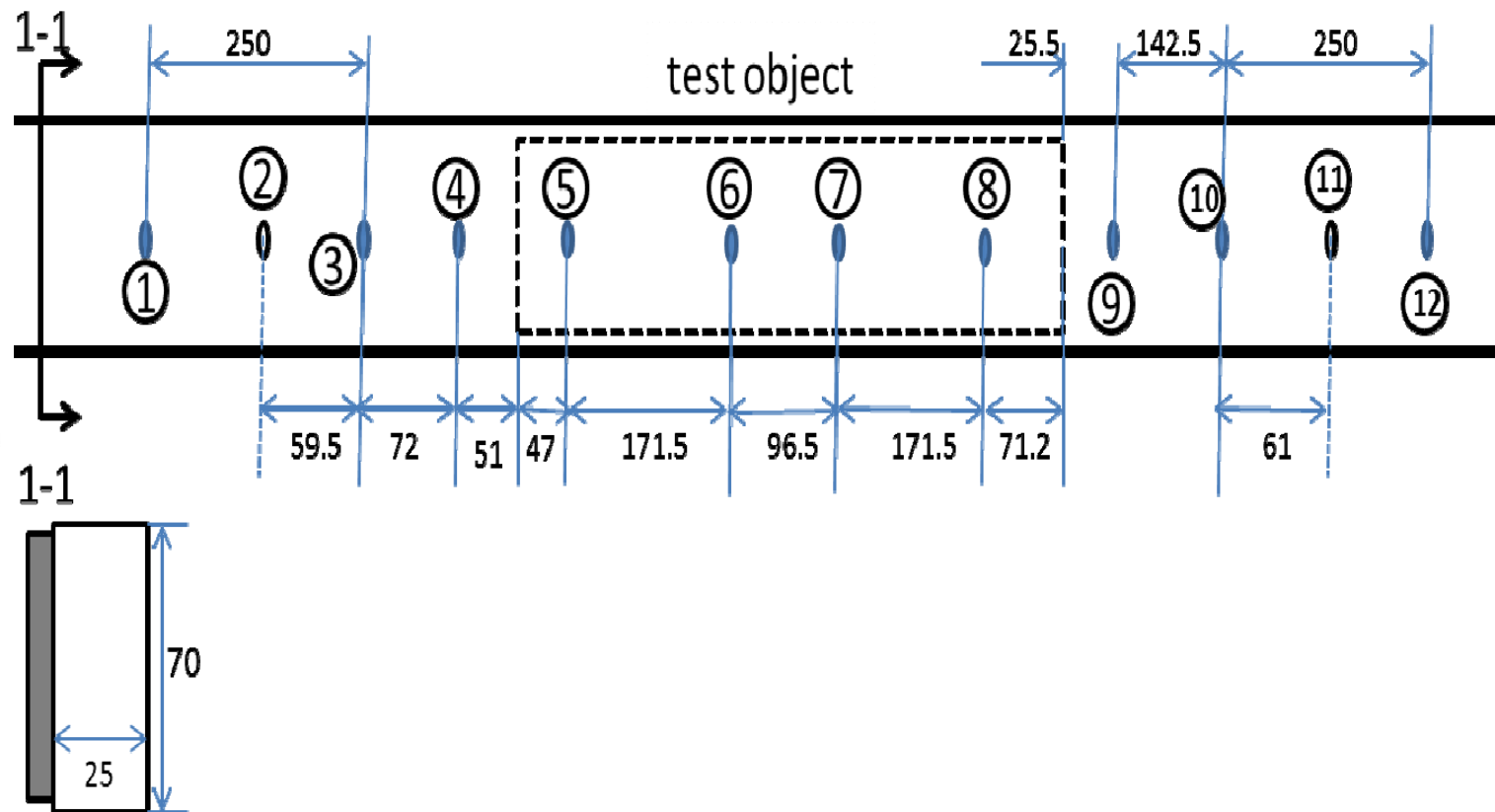
**Calculate the axial wave number assuming no reflection**

**Integrate the pressure and its gradient over the cross section**

**Calculate the impedance**

**Comparison of impedance eduction results using different methods and test rigs. L. Zhou, H. Bodén, C. Lahiri, F. Bake, L. Enghardt and T. Elnady. AIAA/CEAS Aeroacoustics Conference, Atlanta 2014.**

# Microphone positions



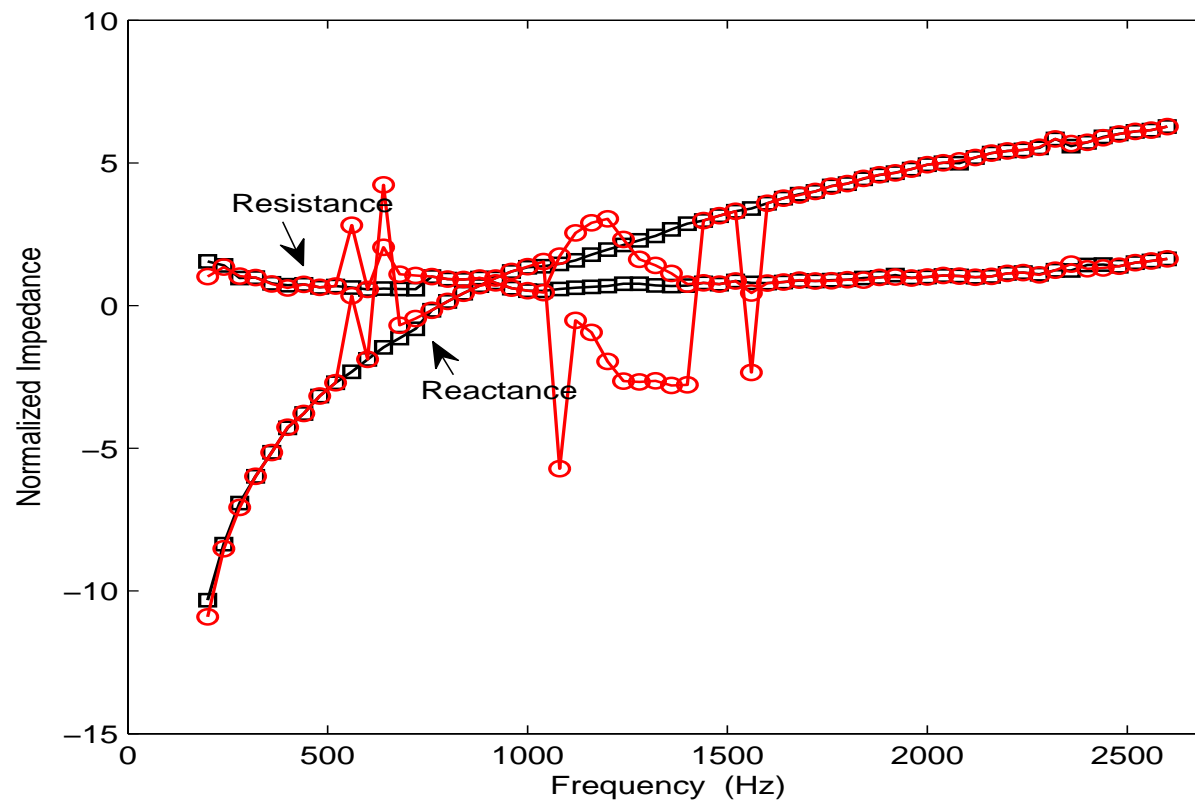


## Impedance Eduction Results

**Black** – only microphones outside the lined section,

**Red** – including three microphones in the lined section

Convected Helmholtz equation, mode matching method



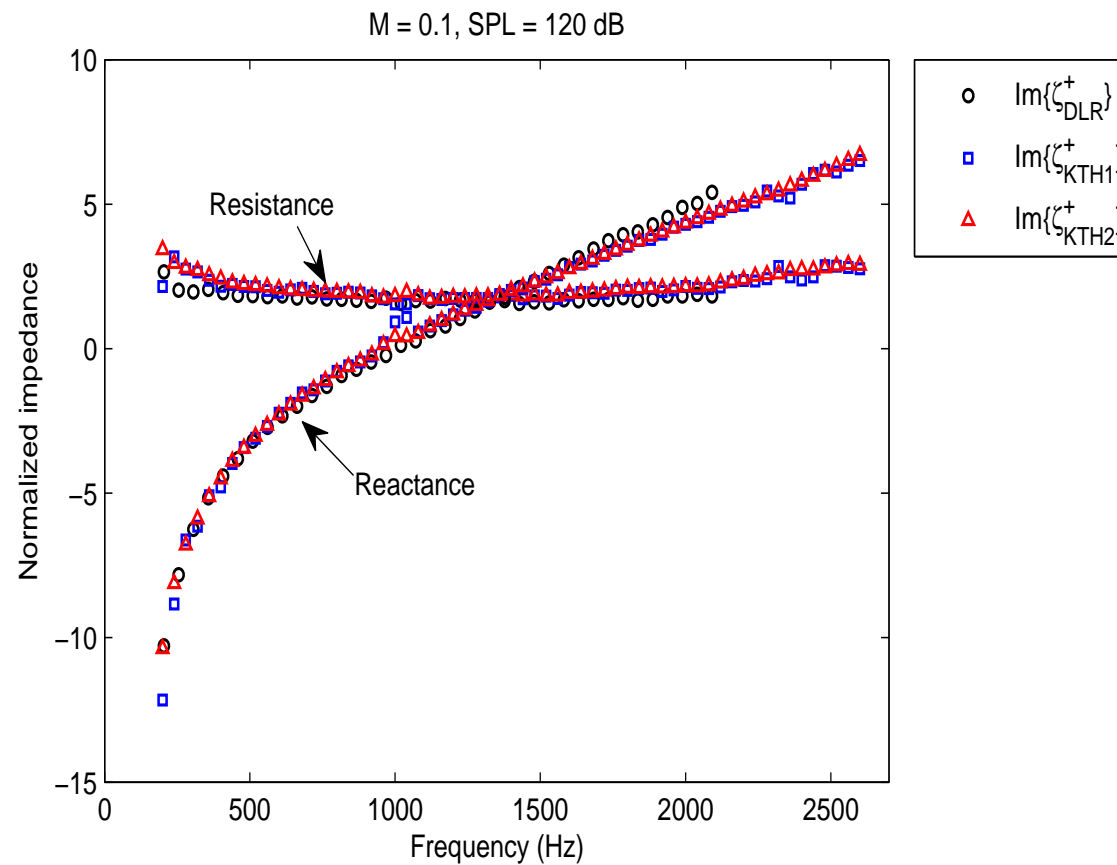
**M = 0**  
**120 dB**

## Impedance Eduction Results

Blue – Convected Helmholtz Equation mode matching method

Red – Strigh forward method

Black – DLR Linus method



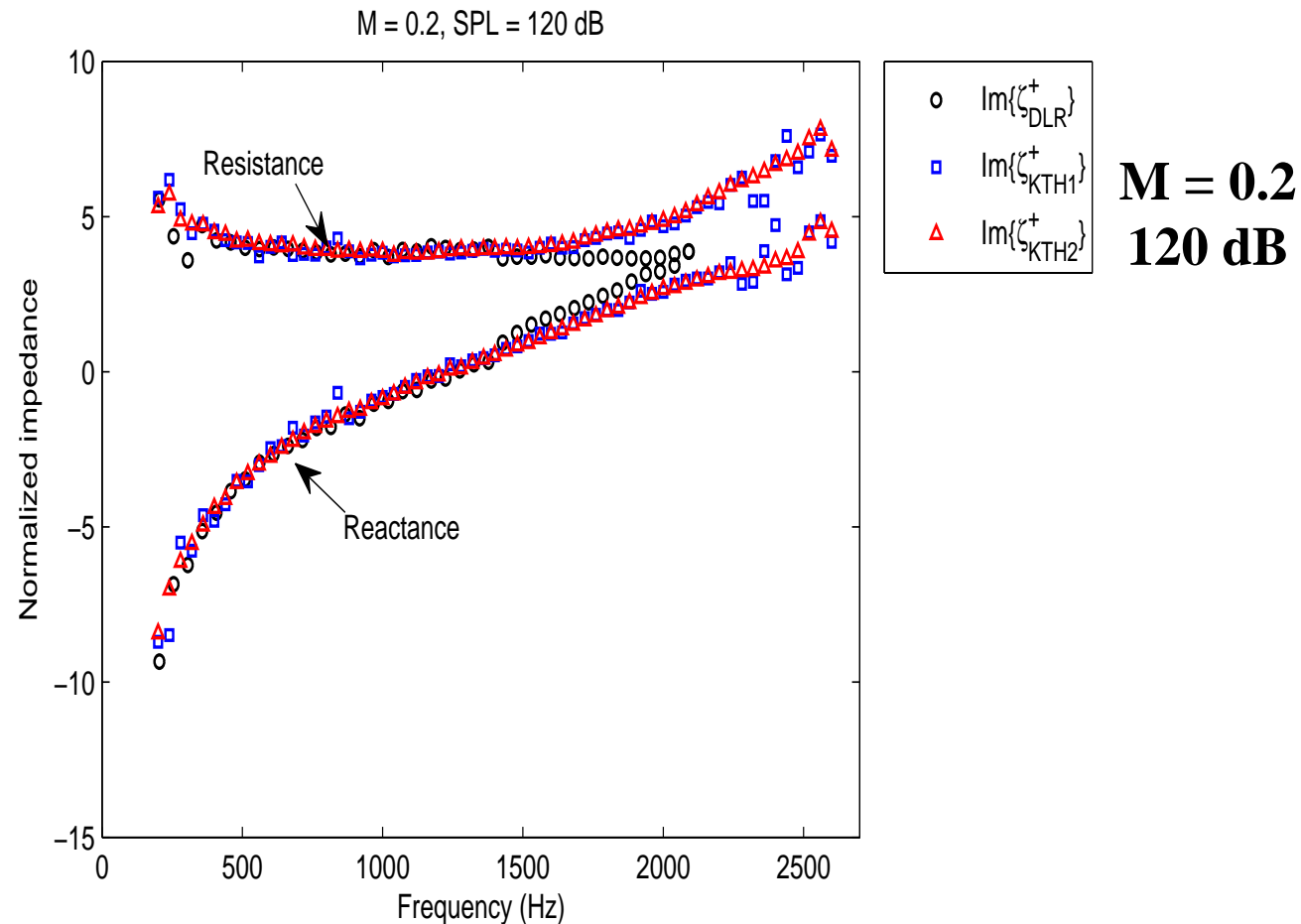
**M = 0.1**  
**120 dB**

## Impedance Eduction Results

Blue – Convected Helmholtz Equation mode matching method

Red – Strigh forward method

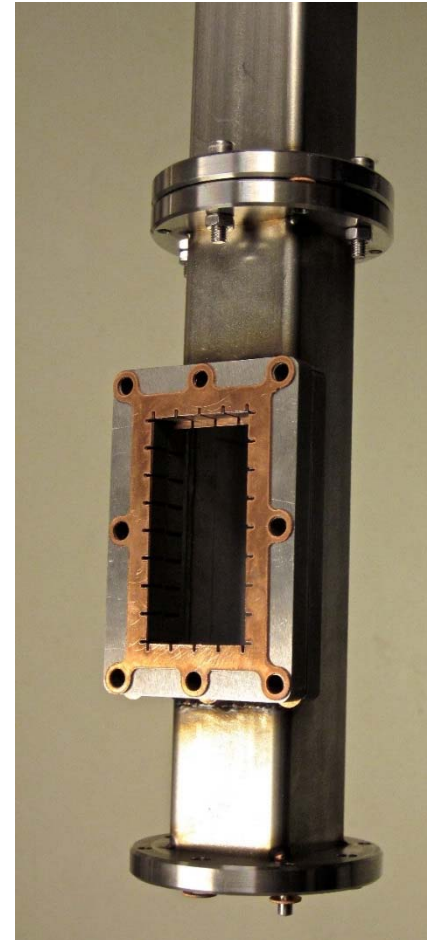
Black – DLR Linus method



## Hot stream liner impedance measurements



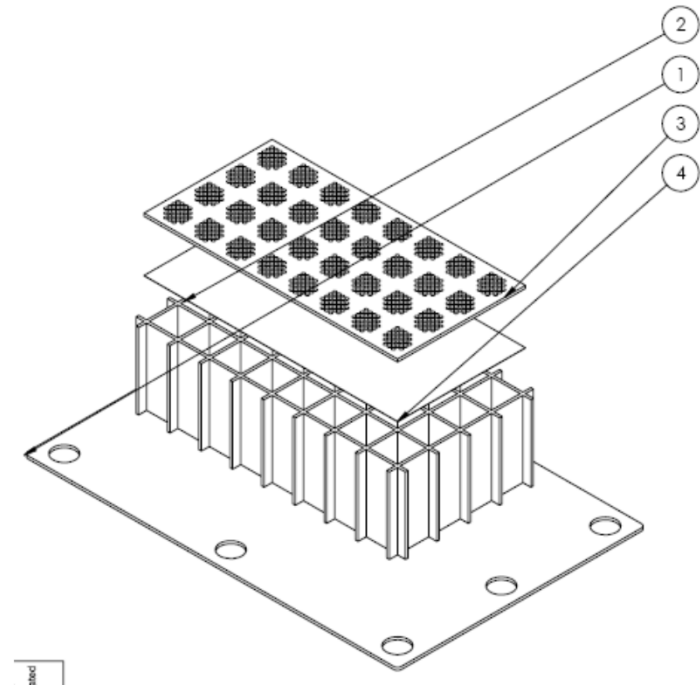
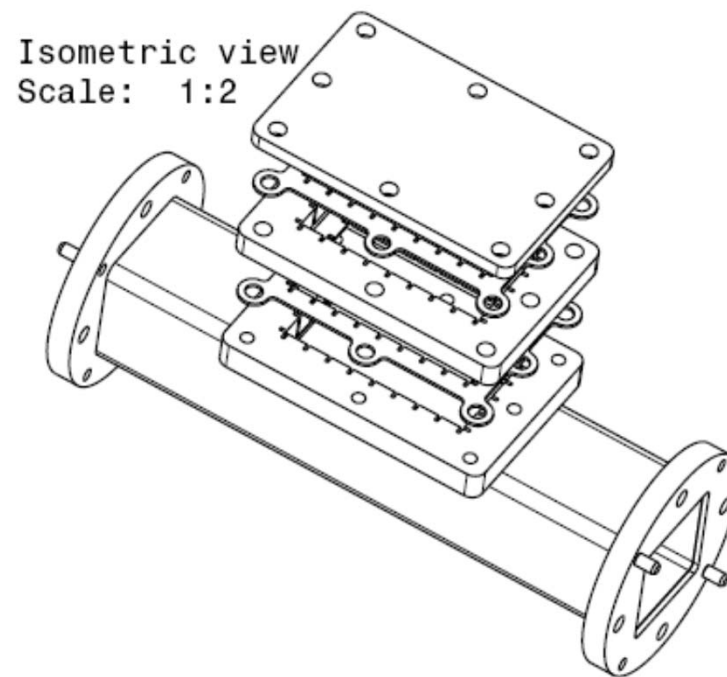
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## Liner Samples for Hot Stream Liner Testing

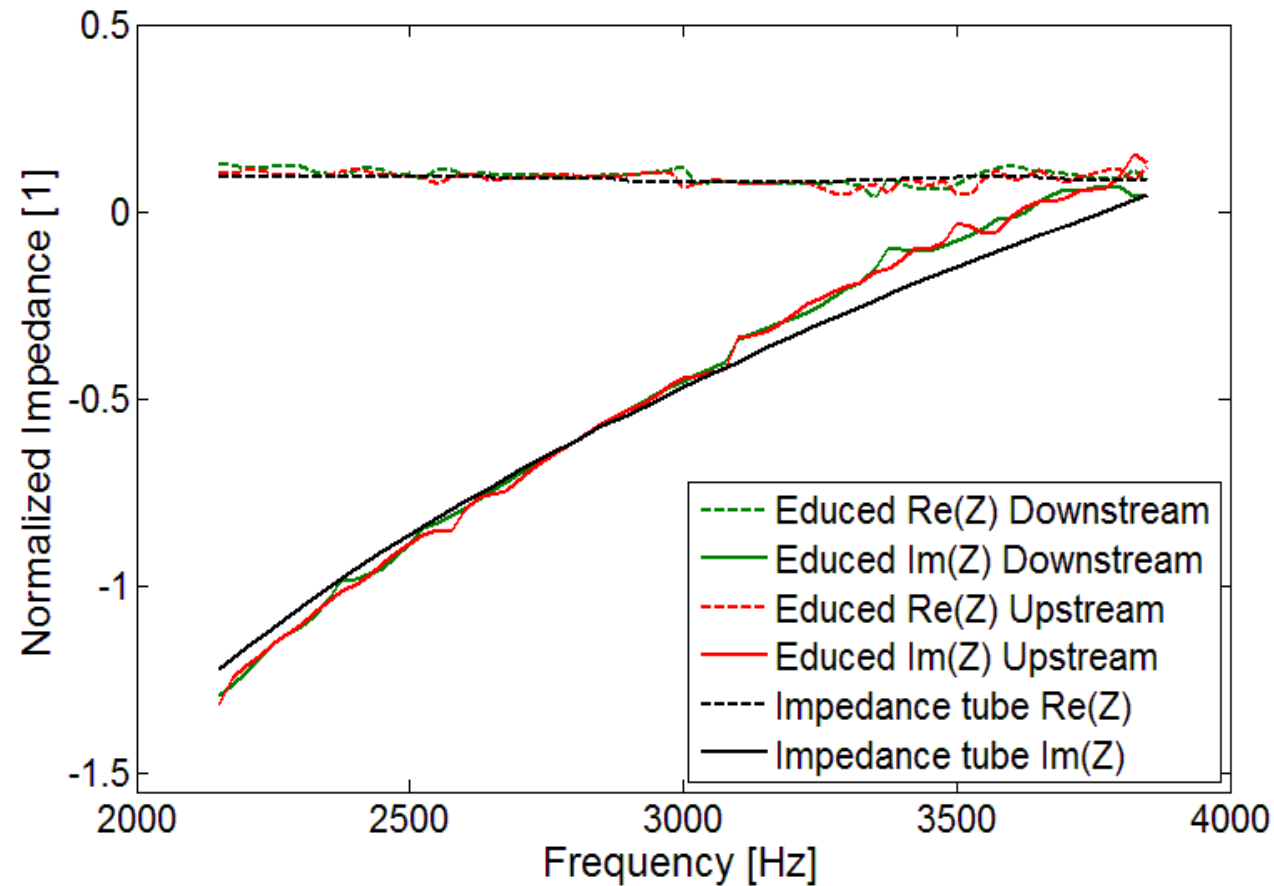


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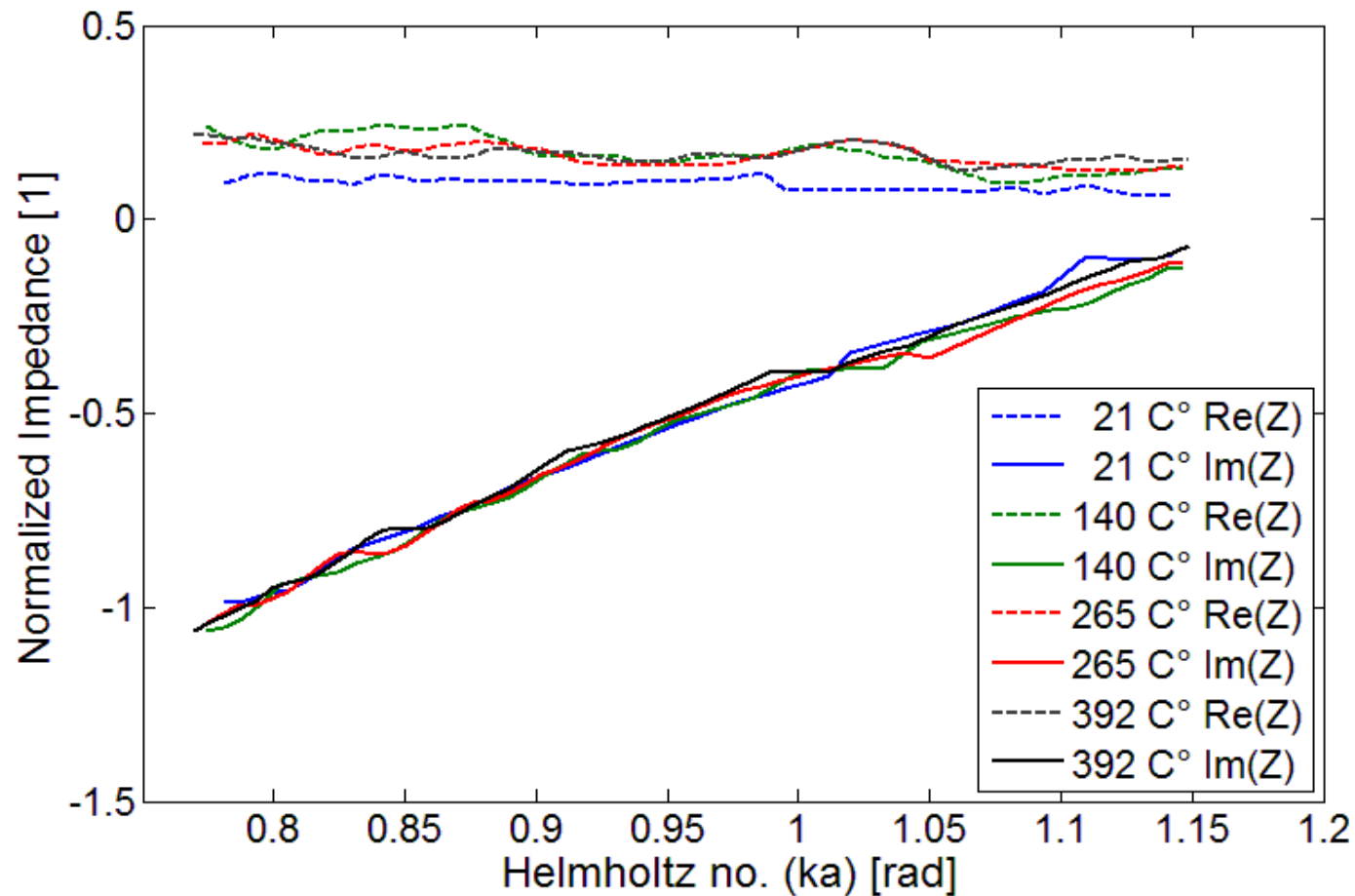
## Impedance Eduction Results

**9 mm cavity, 23 holes, plate thickness 0.6 mm, hole diameter 0.75 mm, porosity 0.20, no foam.**



## Impedance Eduction Results

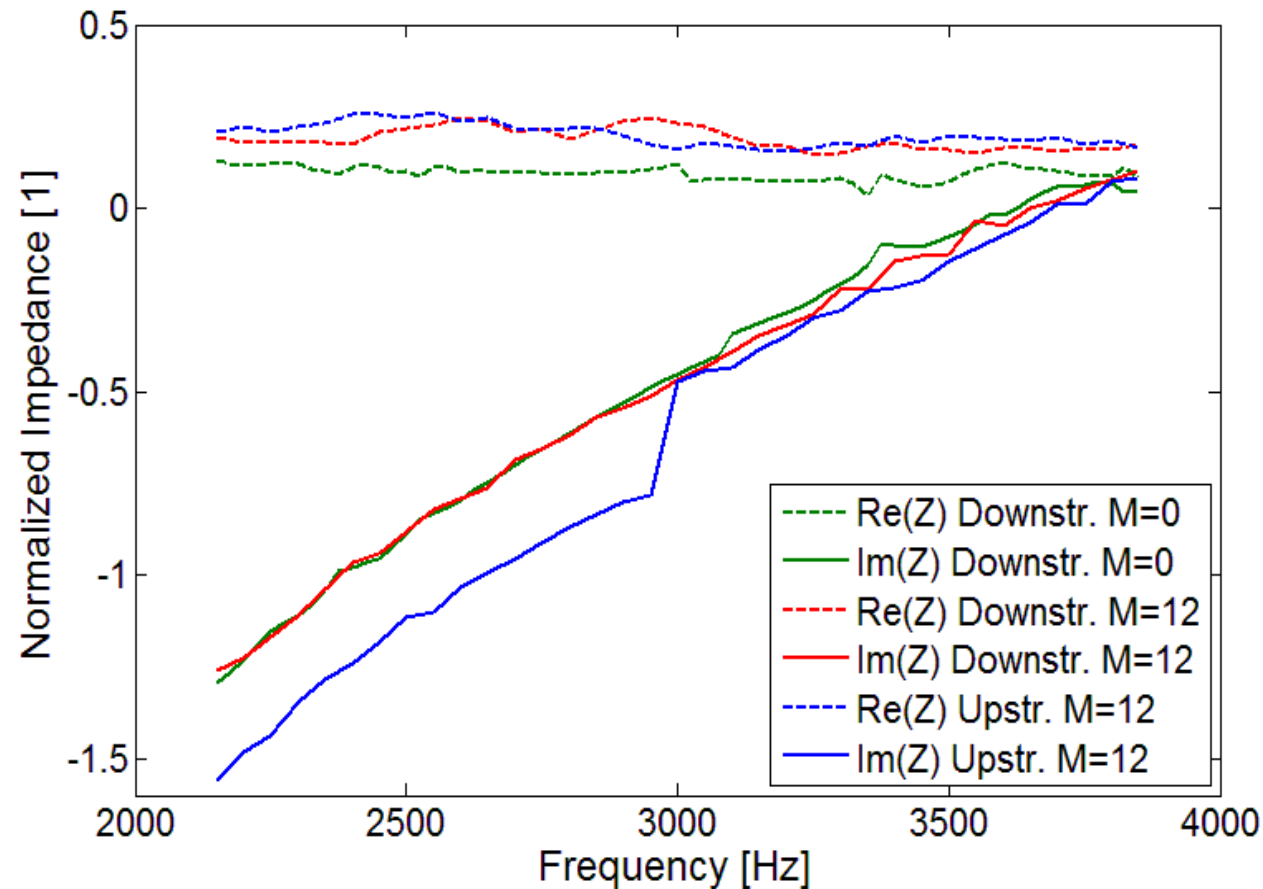
**9 mm cavity, 23 holes, plate thickness 0.6 mm, hole diameter 0.75 mm, porosity 0.20, no foam.**





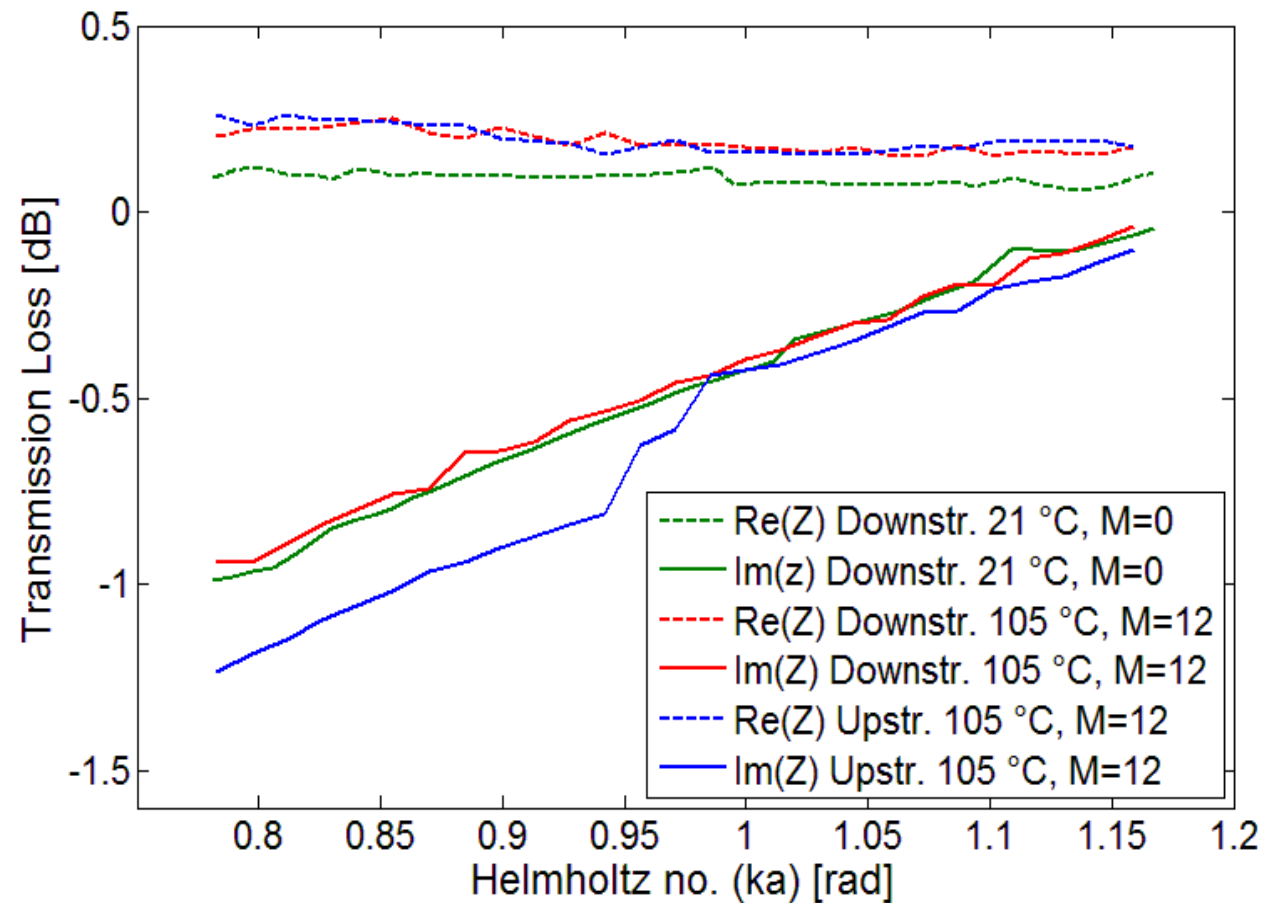
## Impedance Eduction Results

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## Impedance Eduction Results

**9 mm cavity, 23 holes, plate thickness 0.6 mm, hole diameter 0.75 mm, porosity 0.20, no foam.**



# SUMMARY

## The following topics were discussed:

- Passive one-port measurement techniques and the so-called two-microphone method. Sources of errors, the effect of test rig design and flow noise suppression.
- Passive two-port measurement techniques including flow noise suppression methods.
- Application to liner impedance reduction.



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